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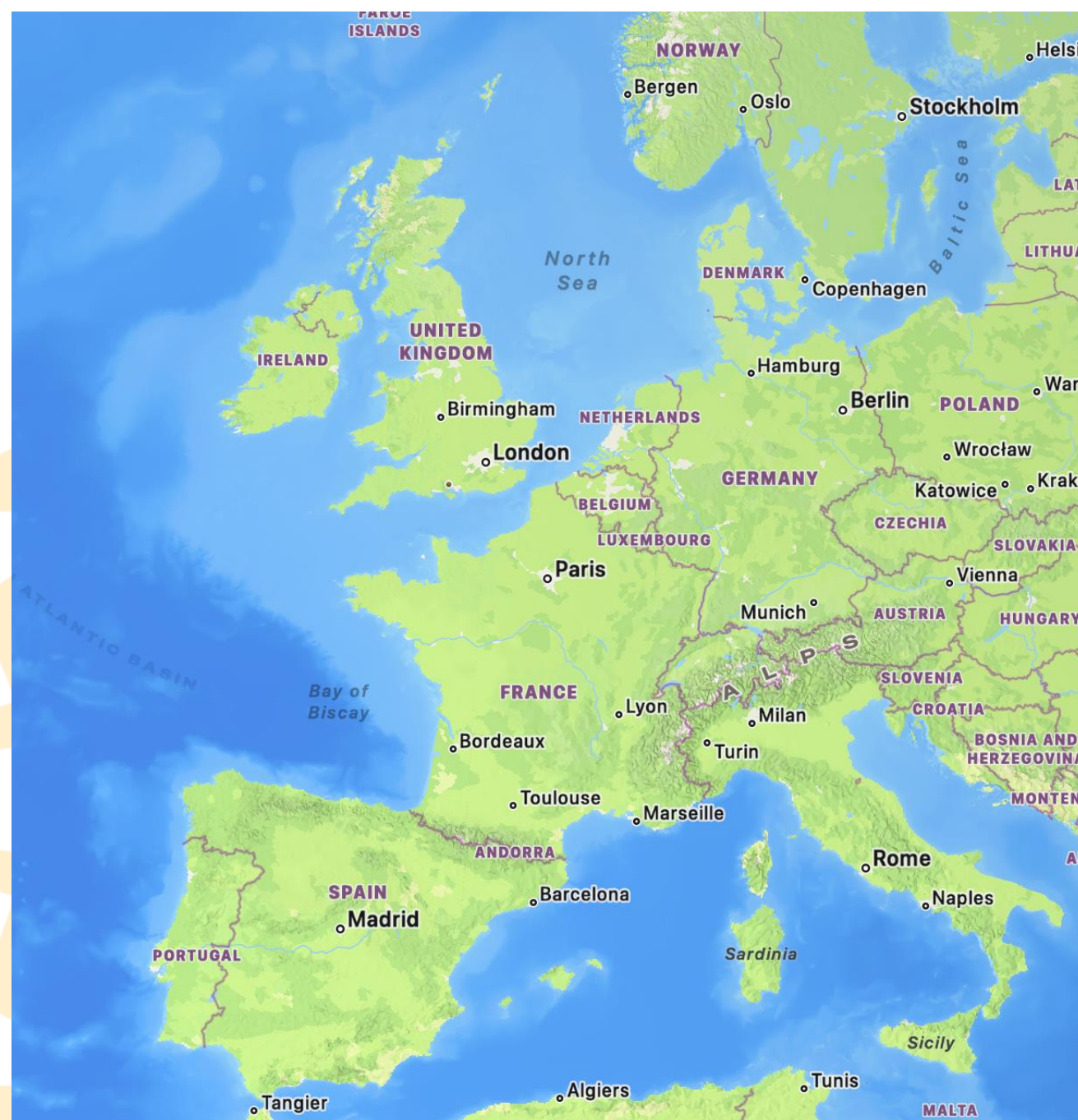


Unravelling the Complexity of Vibroimpact Systems

Daniil Yurchenko

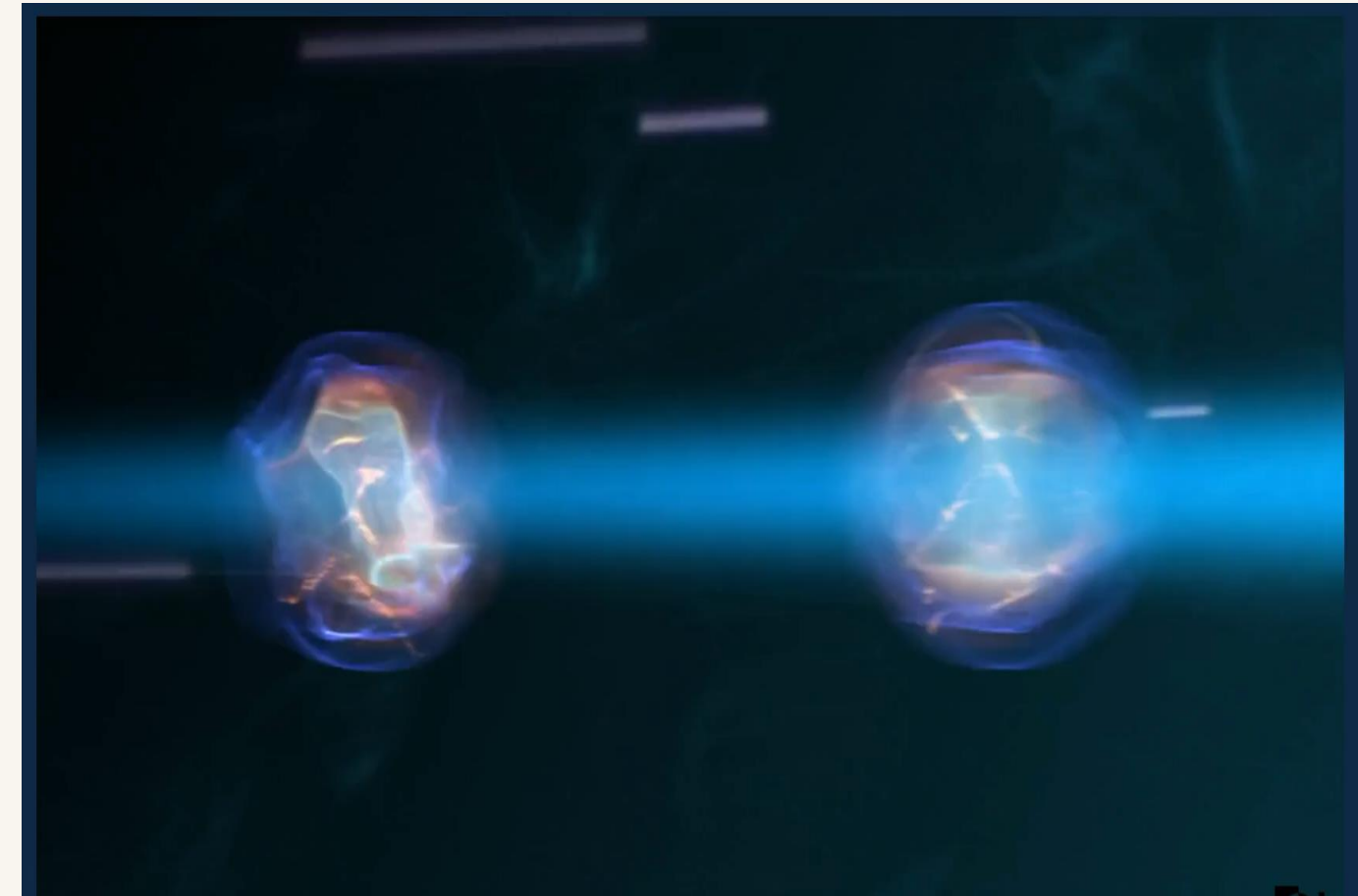
Institute of Sound and Vibration Research, University of Southampton, UK





Content

- Historic overview
- Impact dynamics
- Vibro-impact capsule
- Computer-assisted method
- Conclusion



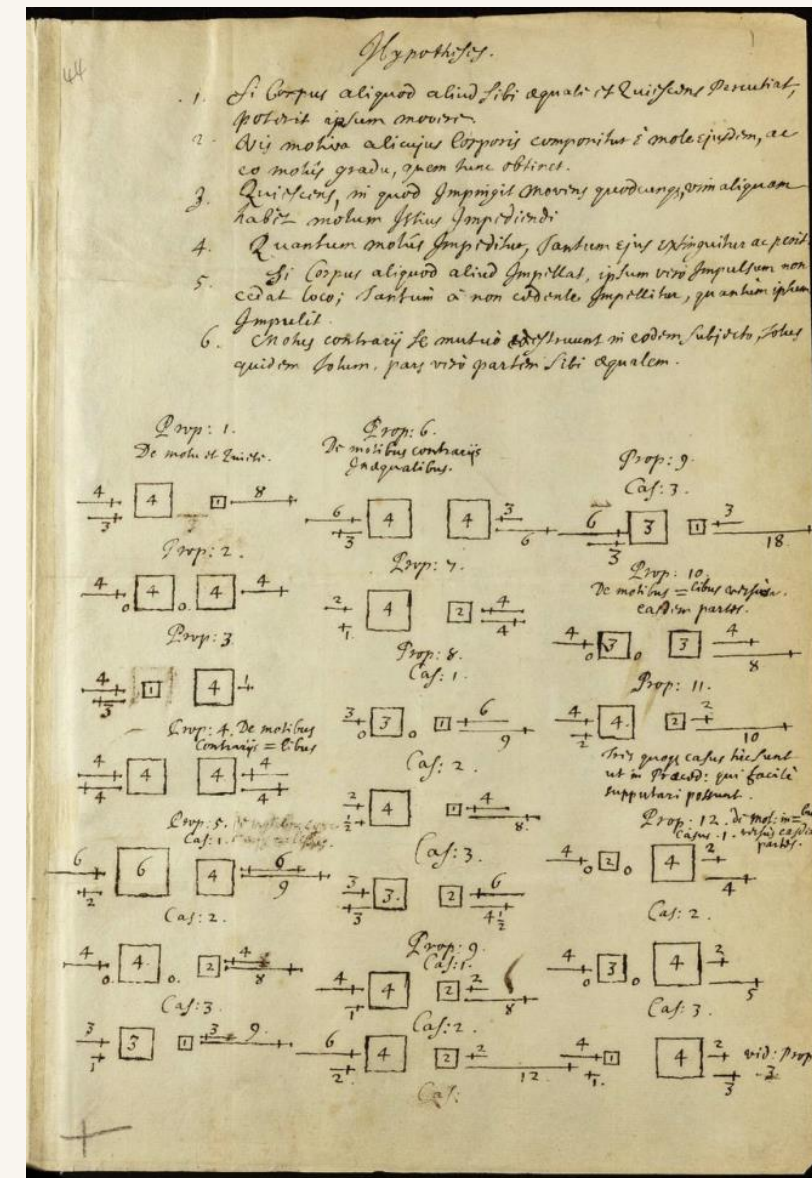
Historic overview

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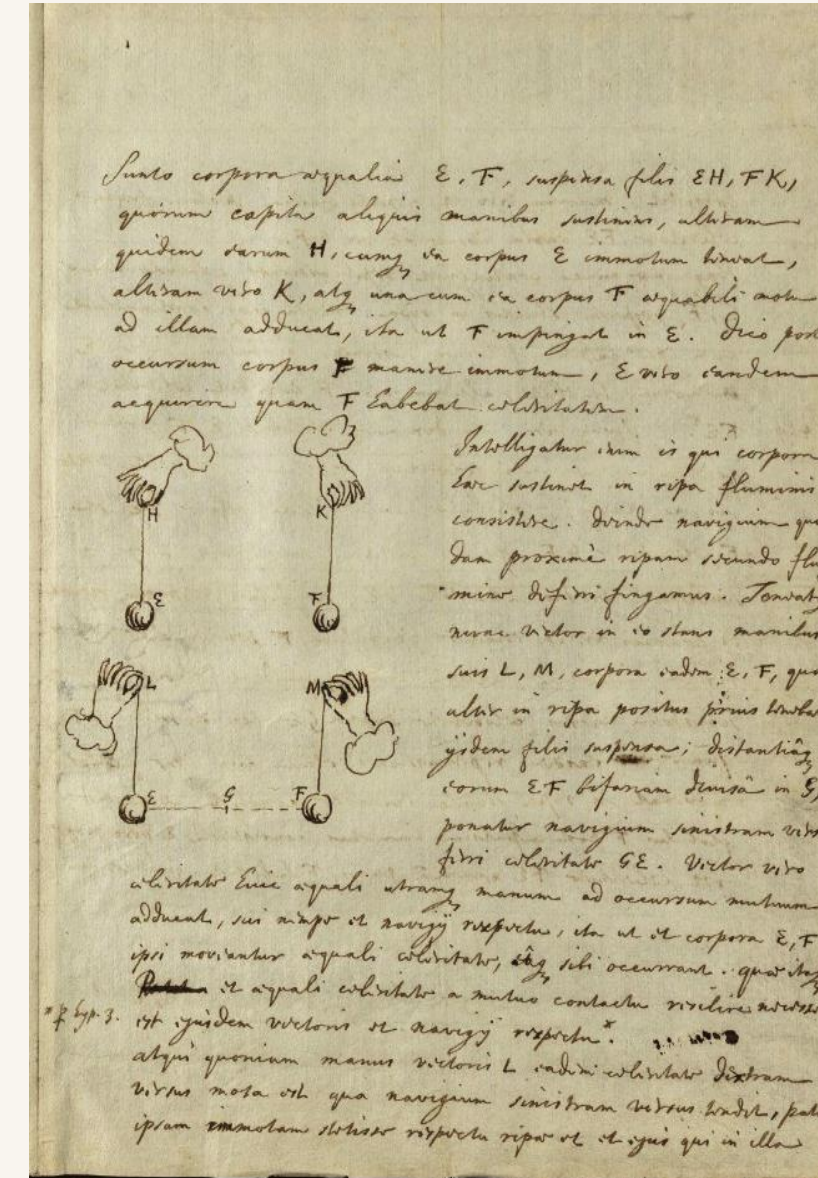
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The law of nature concerning the collision of bodies
C. Wren, 1668



Paper on motion
W. Croone, 1668



On the motion of bodies resulting from impact
C. Huygens, 1668

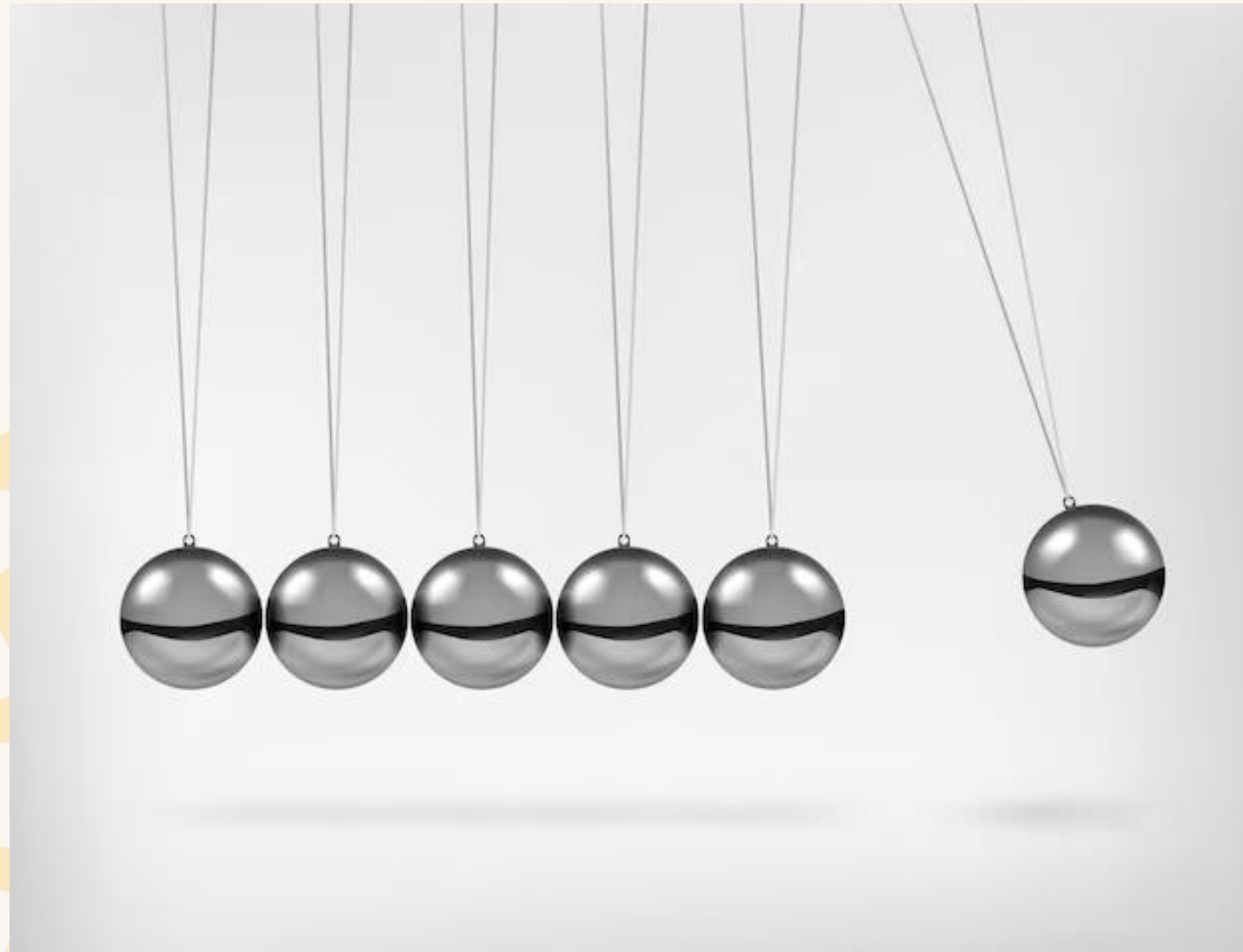
A; and whose ordinate BD will be as the square of the line AB. On the same Laws and Corollaries depend those things which have been demonstrated concerning the times of the vibration of pendulums, and are confirmed by the daily experiments of pendulum clocks. By the same, together with the third Law, Sir Christ. Wren, Dr. Wallis, and Mr. Huygens, the greatest geometers of our times, did severally determine the rules of the congress and reflexion of hard bodies, and much about the same time communicated their discoveries to the Royal Society, exactly agreeing among themselves as to those rules. Dr. Wallis, indeed, was something more early in the publication; then followed Sir Christopher Wren, and, lastly, Mr. Huygens. But Sir Christopher Wren confirmed the truth of the thing before the Royal Society by the experiment of pendulums, which Mr. Mariotte soon after thought fit to explain in a treatise entirely upon that subject. But to bring this experiment to an accurate agreement with the theory, we are to have a due regard as well to the resistance of the air as to the elastic force of the concurring bodies. Let the spherical bodies A, B be suspended by the parallel and equal strings AC, BD, from the centres C, D. About these centres, with those intervals, describe the semicircles EAF, GBH, bisected by the radii CA, DB. Bring the body A to any point R of the arc EAF, and (withdrawing the body B) let it go from thence, and after one oscillation suppose it to return to the point V: then RV will be the retardation arising from the resistance of the air. Of this RV let ST be a fourth part, situated in the middle, to wit, so as RS and TV may be equal, and RS may be to ST as 3 to 2 then will ST represent very nearly the retardation during the descent from S to A. Restore the body B to its place: and, supposing the body

The Principia (I. Newton, 1686)
Translated by A Motte
1846

Historic overview

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Newton cradle

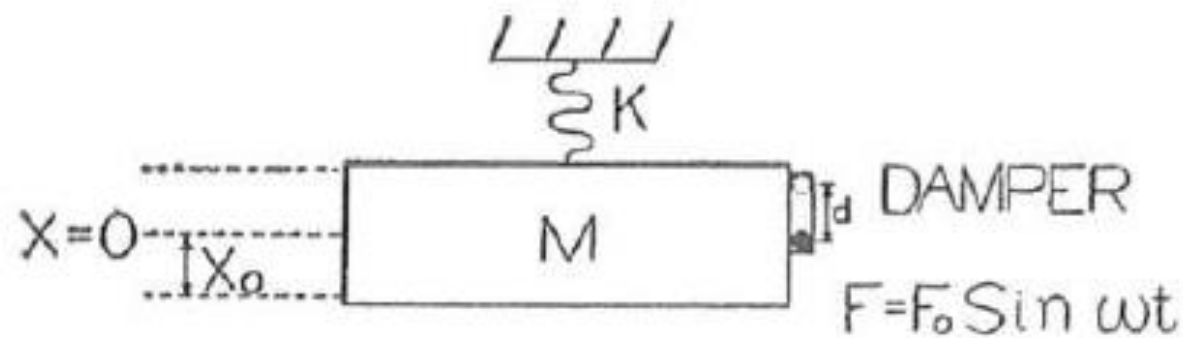


Invented by Edme Mariotte

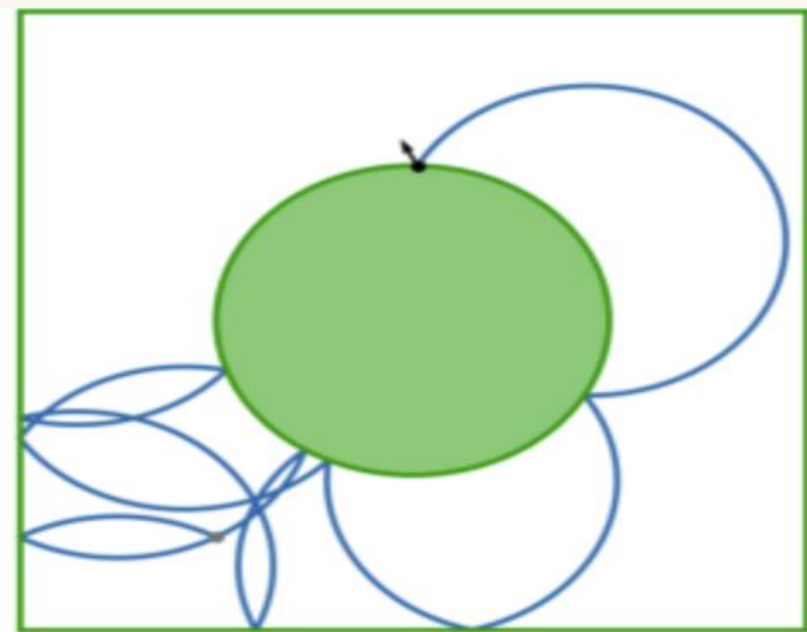


P. Lieber and D.P. Jensen; 1945

This paper considers some theoretical aspects of the acceleration damper itself and gives a formula enabling one to use empirical damping data in theoretical investigations of flutter and vibration phenomena. The elastic rebound between the mass particle and its container is assumed to be zero.



Sinai billiard; 1963



Vibro-impact systems have rich dynamical behavior

- Chaotic motion
- Possesses classical and non-classical bifurcations, including Symmetry breaking, Grazing, Border-Collision and Chattering bifurcations
- Use special methods developed for VI systems (Zhuravlev's transformation, Mapping approach)
- Motivated targeted energy transfer and nonlinear energy sinks

Impacting systems

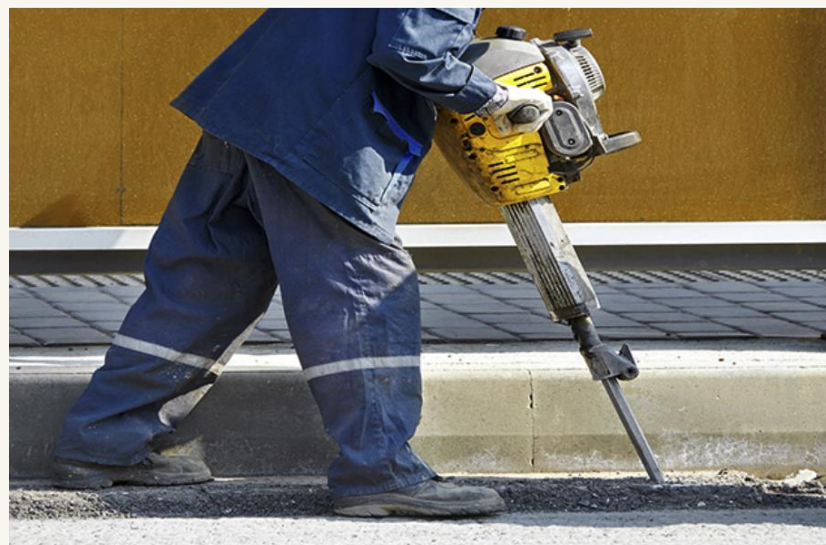
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Basketball, Baseball, Boules, Bowlin, Cricket, Croquet, Billiard,
Football, Golf, Bandy Field Hockey, Netball, Polo, Table football, Table
Tennis, Tennis, Volleyball and many others

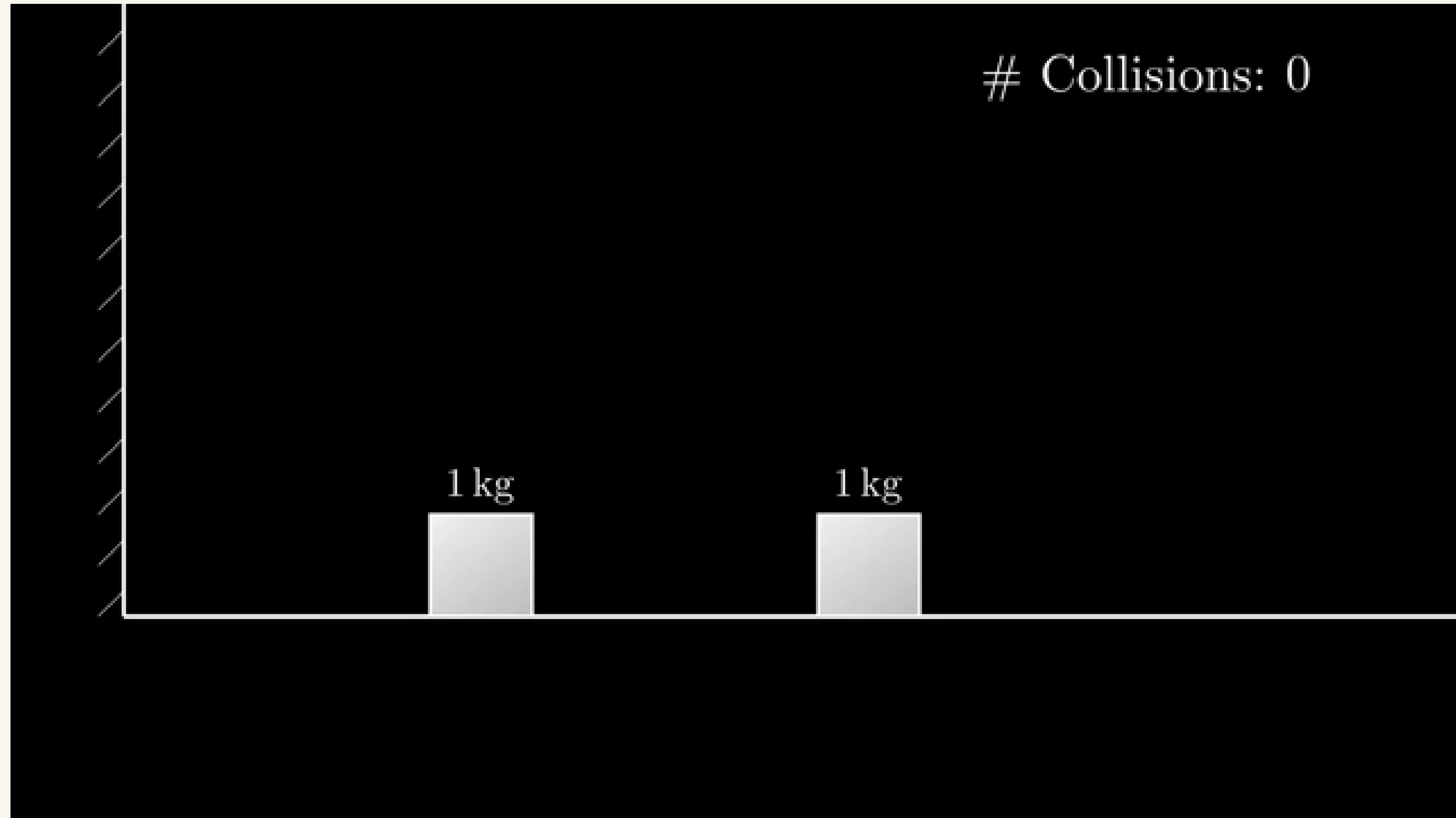
Impacting systems

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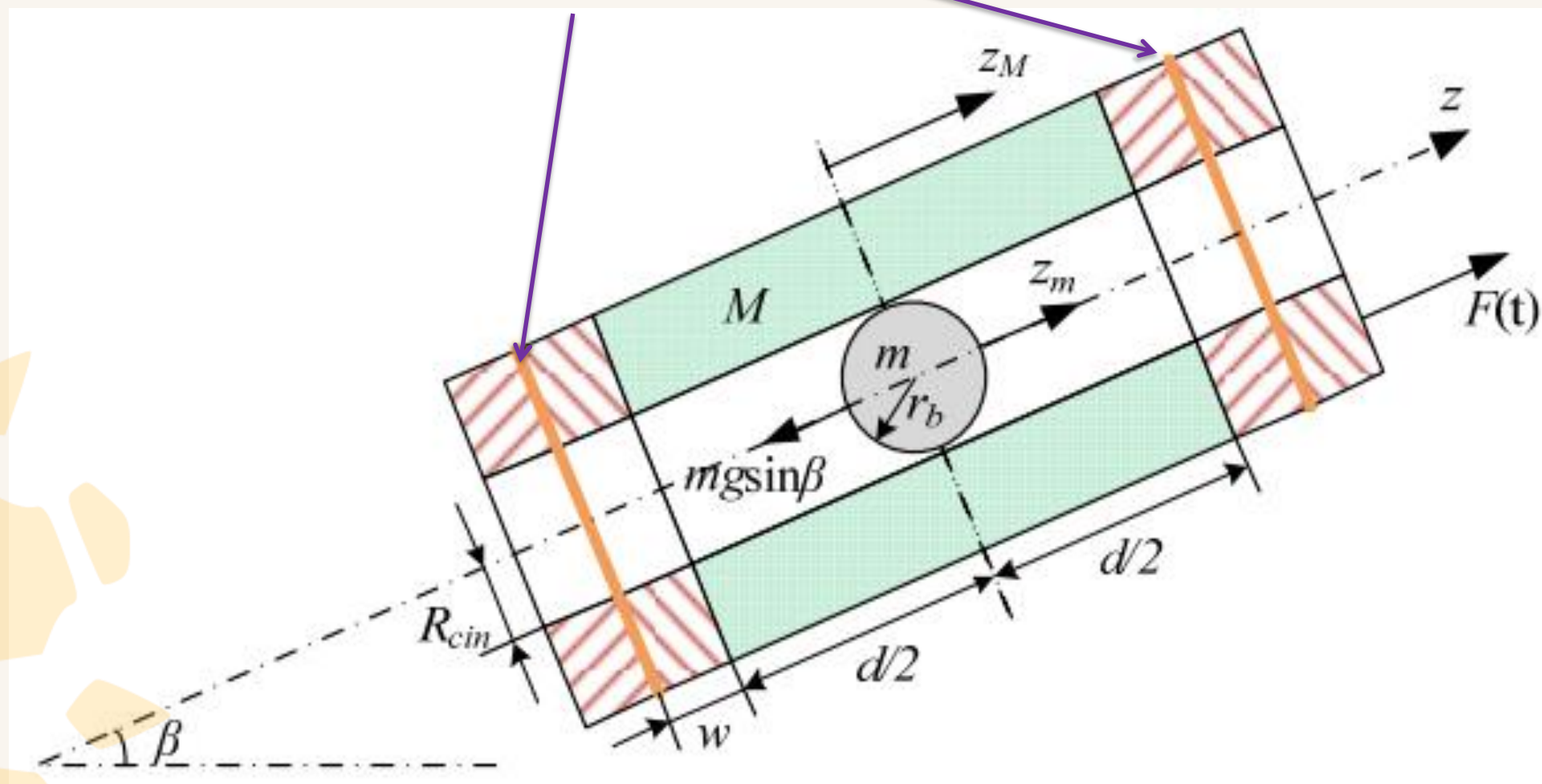


Impacting systems

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DE membranes



No damping and no friction are assumed

Motion of the tube is:

$$z_M''(t) = \frac{A \cos(2\pi f_0 t)}{M}$$

Motion of the ball is:

$$z_m''(t) = -g \sin \beta$$

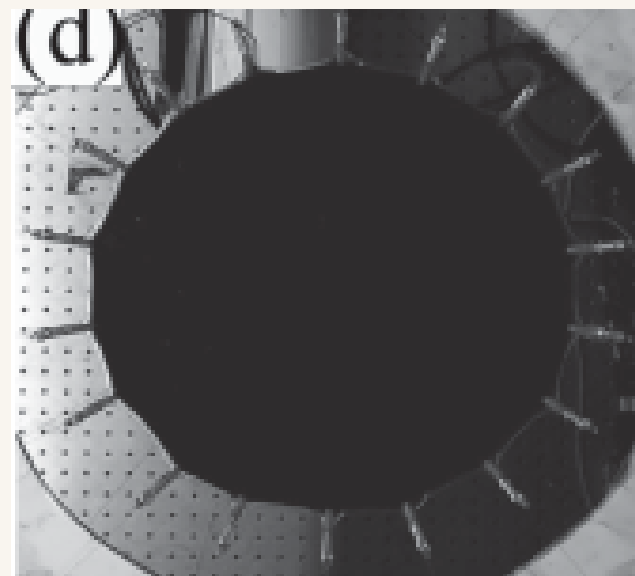
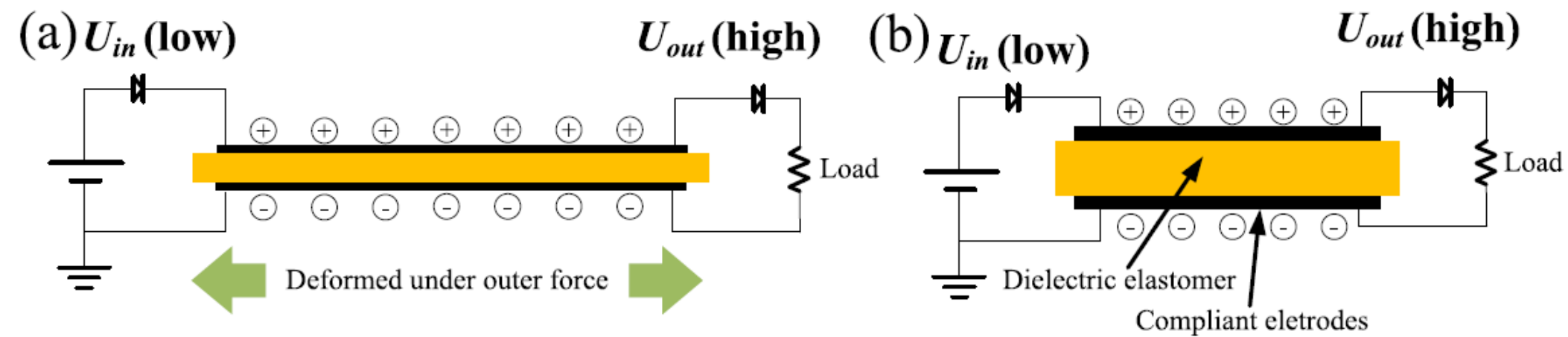
$$d = \frac{sM\omega^2}{A\pi^2}$$

The impact conditions are:

$$\Delta z = z_M - z_m = s/2 \quad \text{for impacts on the left}$$

$$\Delta z = z_M - z_m = -s/2 \quad \text{for impacts on the right}$$

$$Q = CV, C = \frac{\epsilon\epsilon_0 Vol}{z^2}, dQ = VdC$$



Maps approach

$$P_1 : \partial B \mapsto \partial T, \quad P_2 : \partial T \mapsto \partial B, \\ P_3 : \partial B \mapsto \partial B, \quad P_4 : \partial T \mapsto \partial T.$$

We consider 1:1 motion

$$P_1 P_2$$

We consider 2:1 motion

$$P_1 P_2 P_3$$

To understand the conditions that lead to the higher efficiency and using the design parameters to sustain this motion

Attracting (stable) behavior for $r < 1$

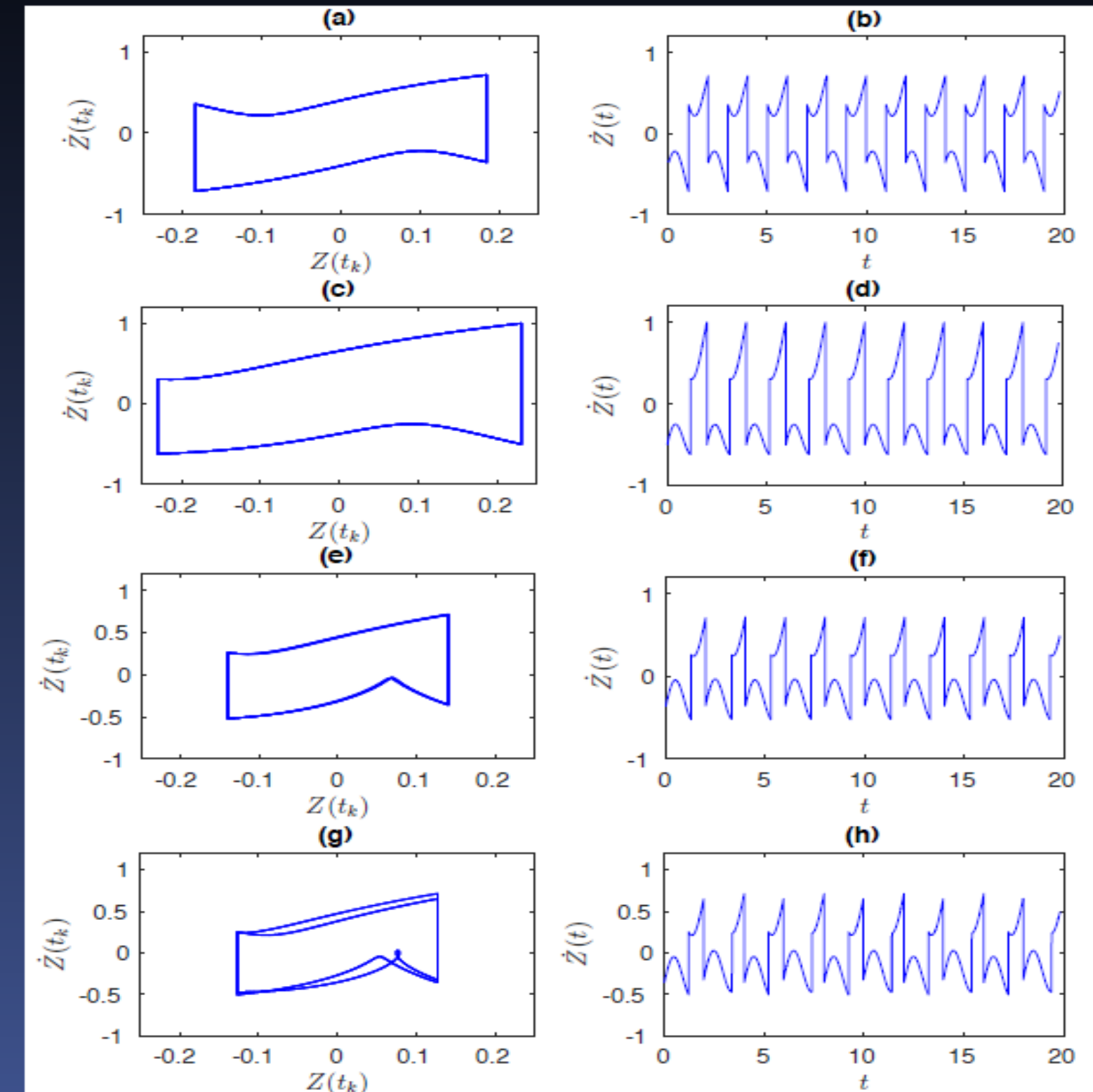
Symmetric

$$\beta = 0, \|\hat{F}\| = 3$$

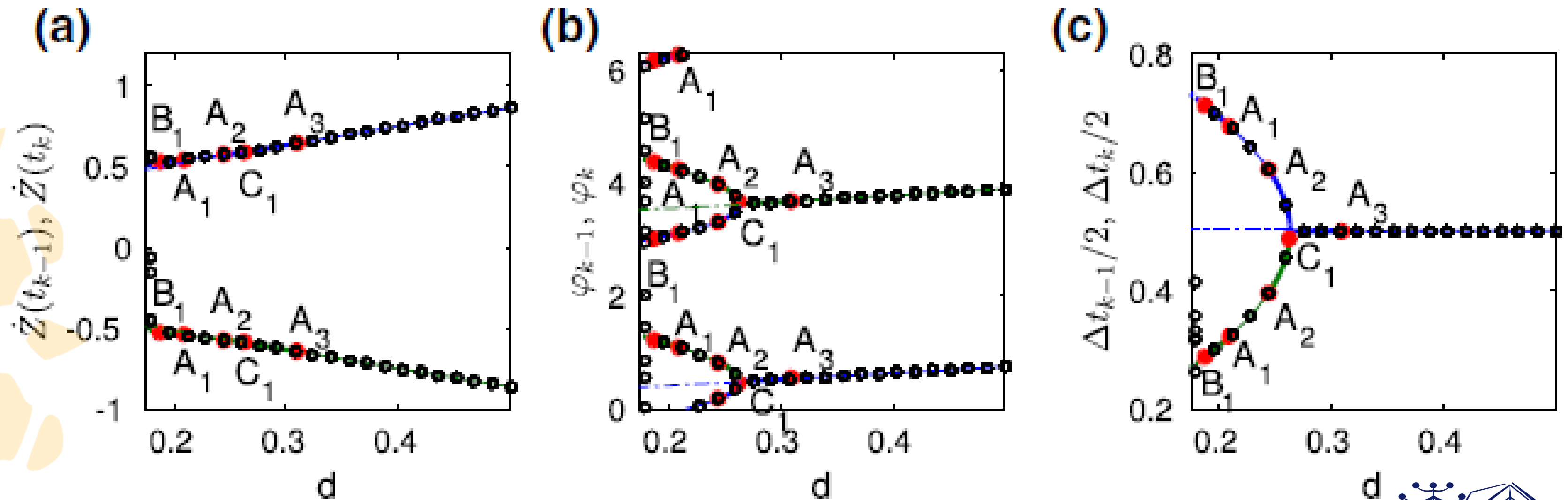
asymmetric T-periodic

$$\beta = \pi/4$$

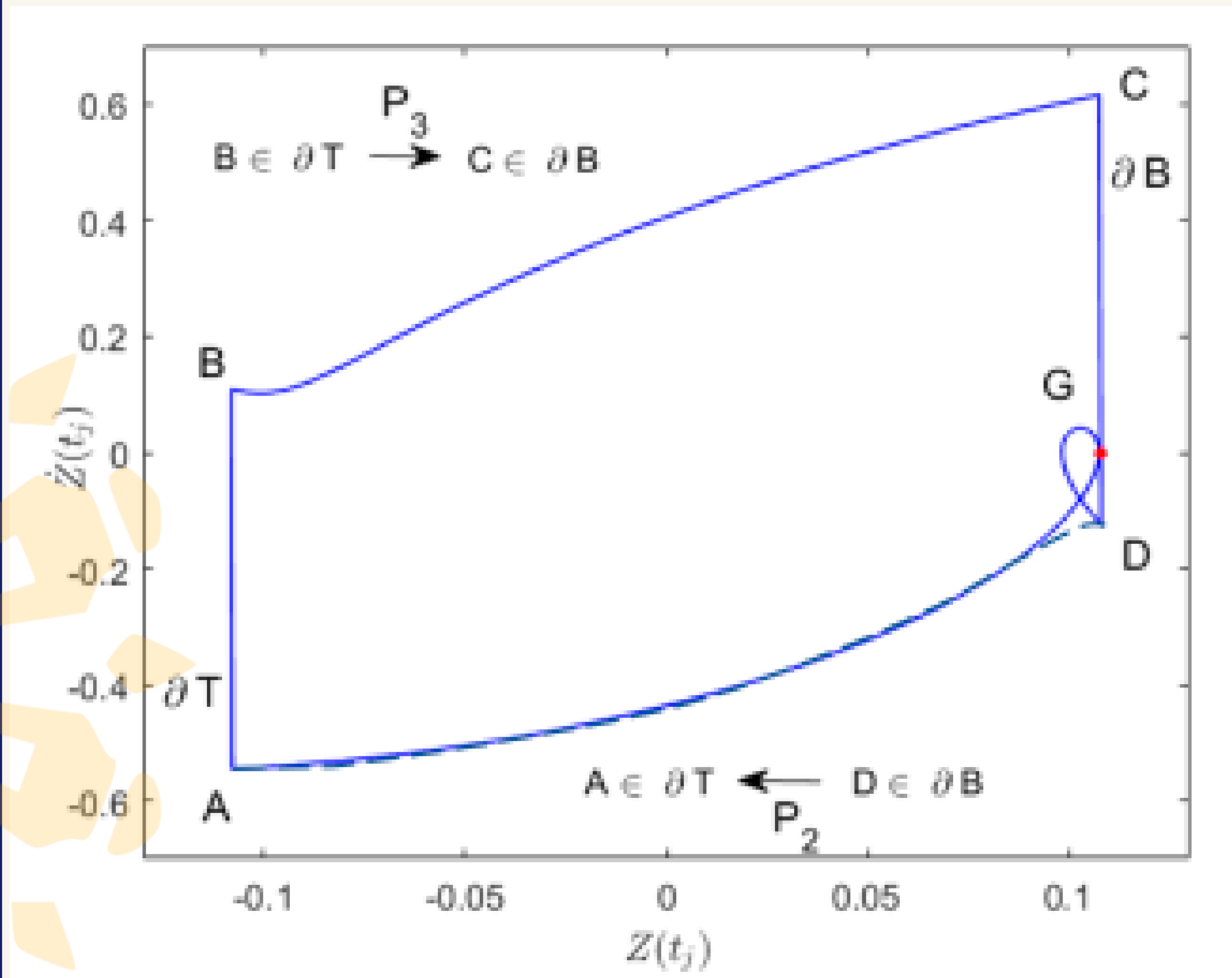
asymmetric 2T-periodic



Classical bifurcations, which can be identified from linear analysis, help us to predict motion stability



Grazing bifurcations



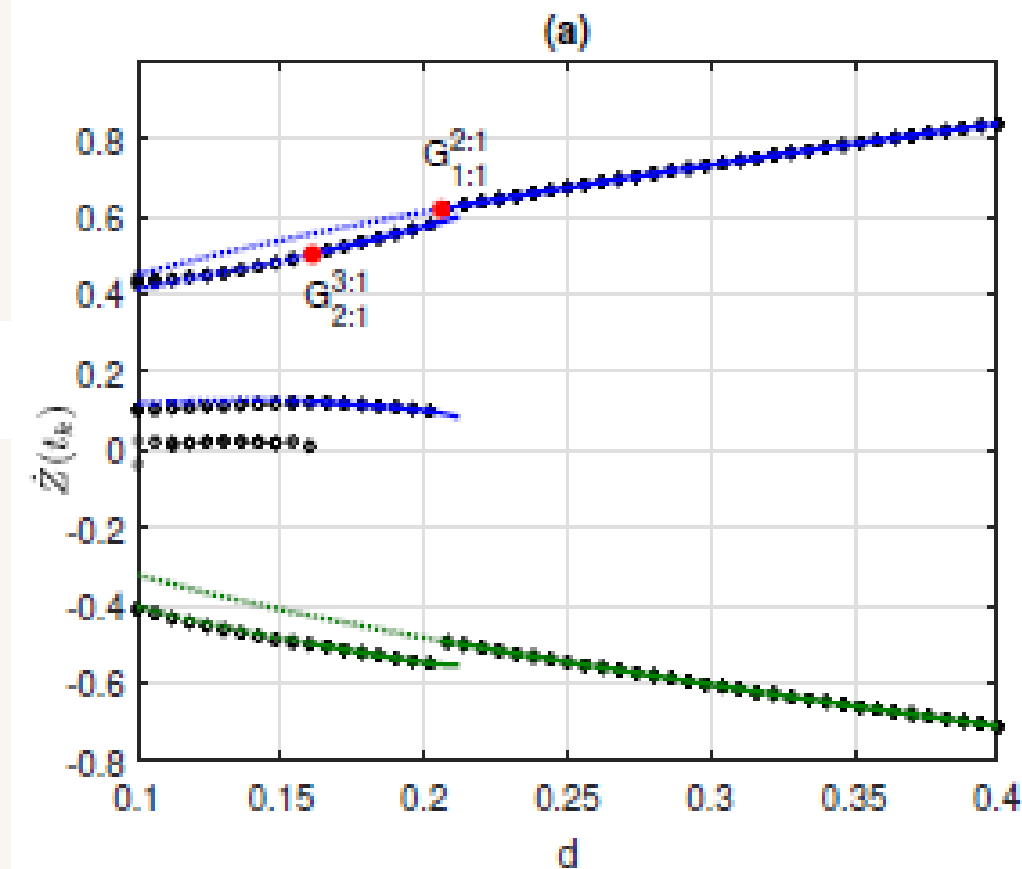
Using the maps and conditions

$$\dot{Z}(t_G) = 0, Z(t_G) = -d/2$$

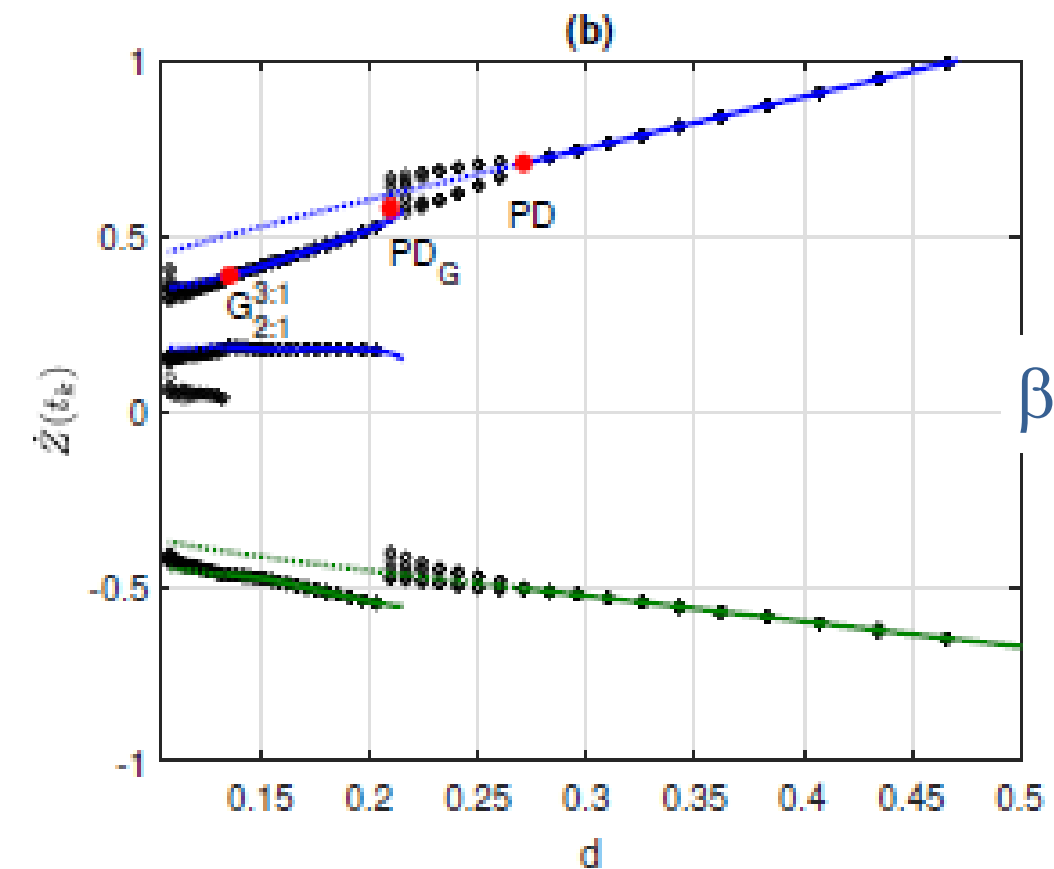
It is possible to establish the exact set of parameters for grazing.

Unlike PD and FB, the linear stability analysis cannot predict grazing bifurcation

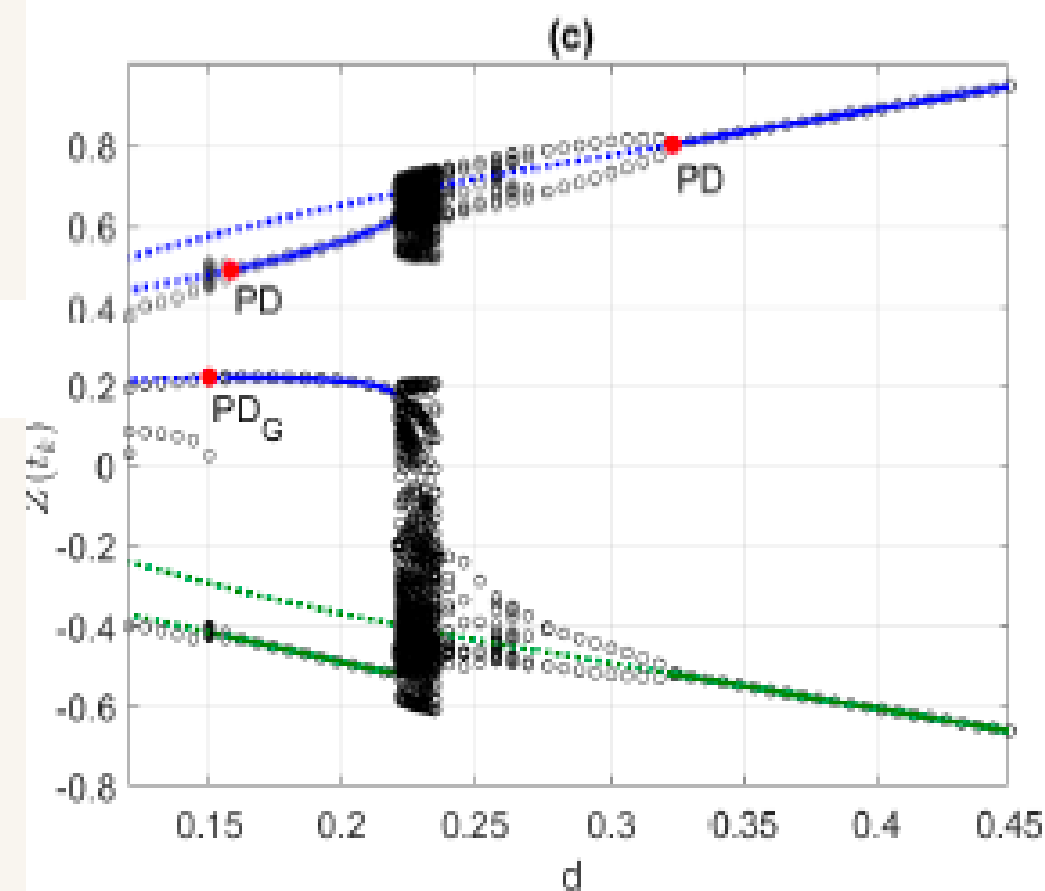
$\beta=\pi/9, r=0.3, A=5N$



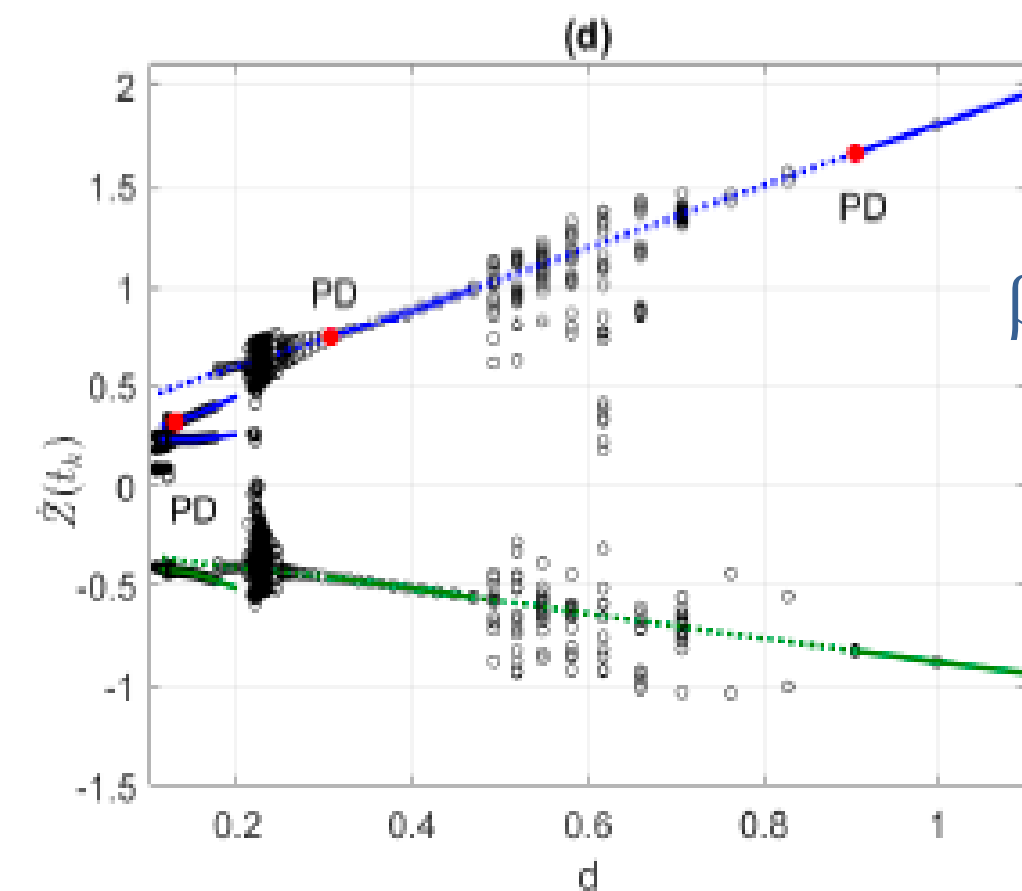
$\beta=\pi/4, r=0.5, s=0.5m$

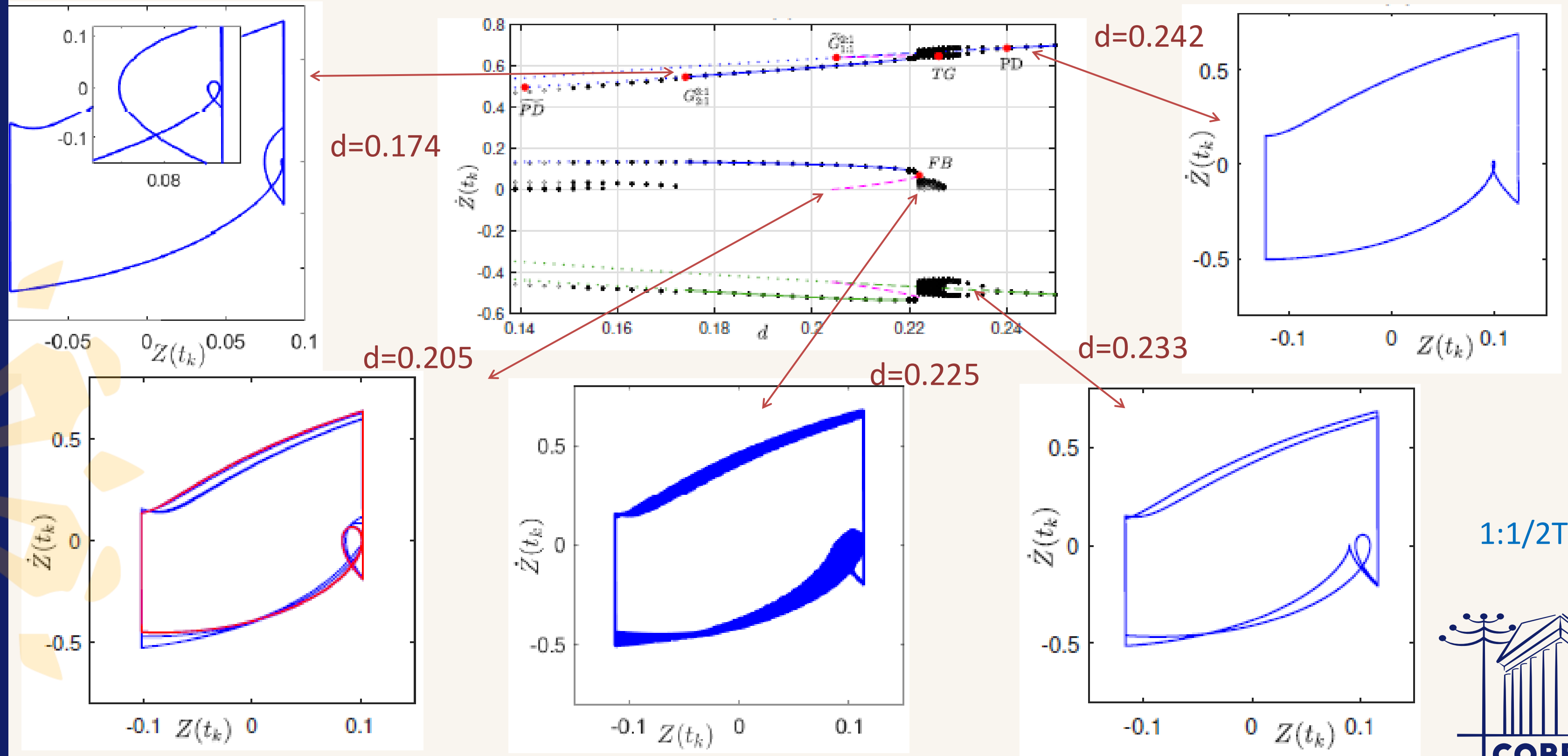


$\beta=\pi/3, r=0.5, A=5N$

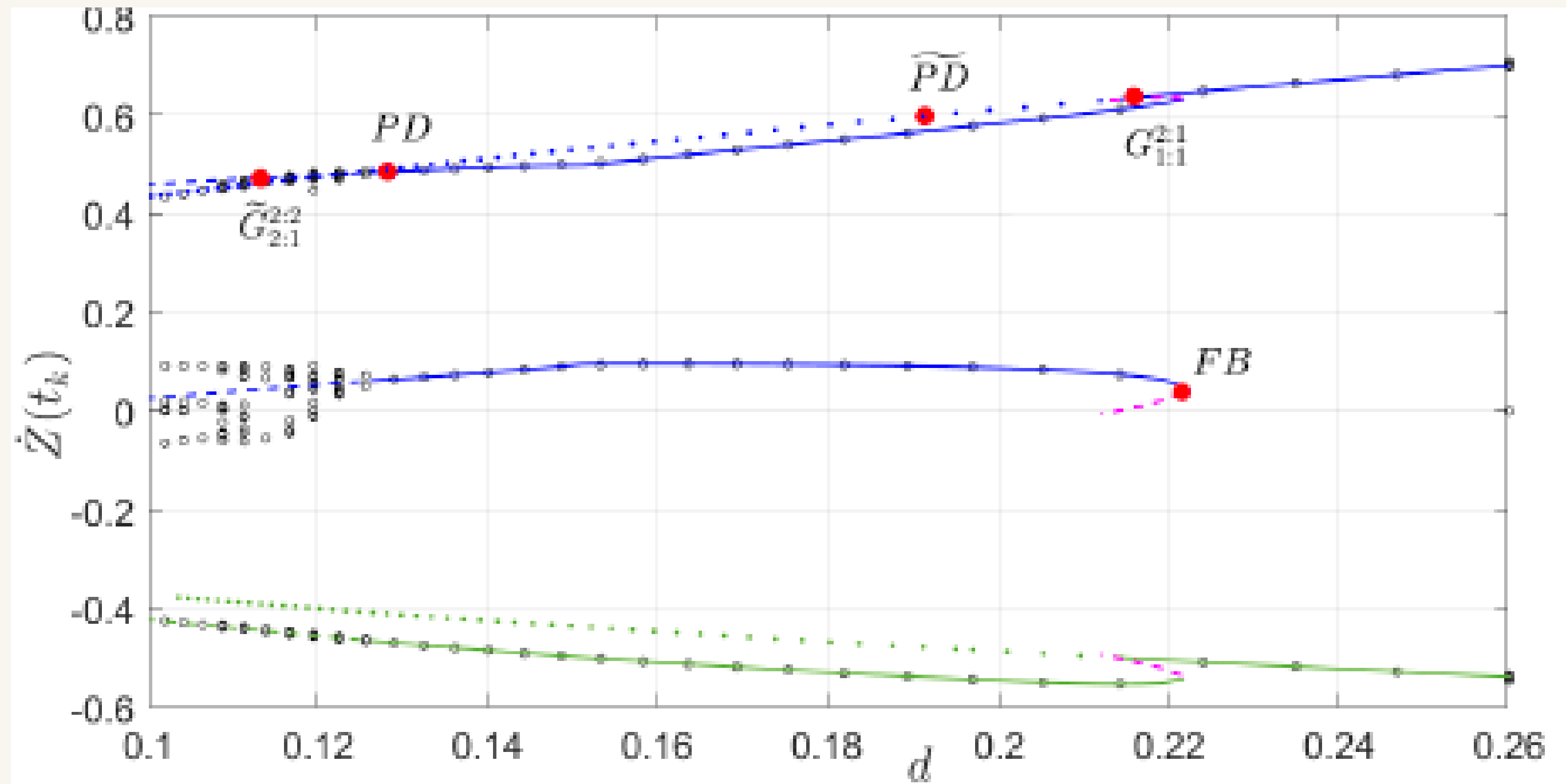


$\beta=\pi/2, r=0.7, s=0.5m$

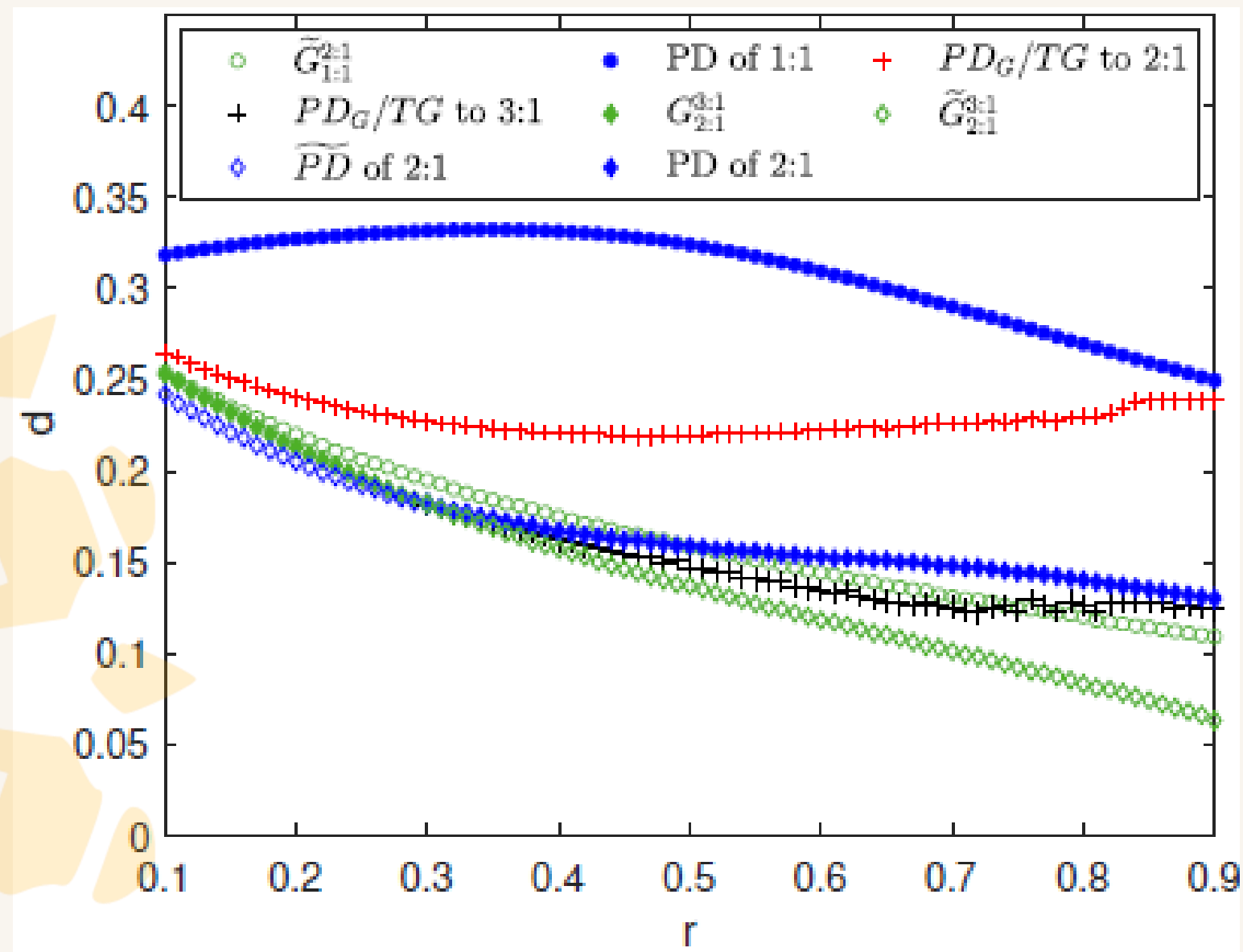




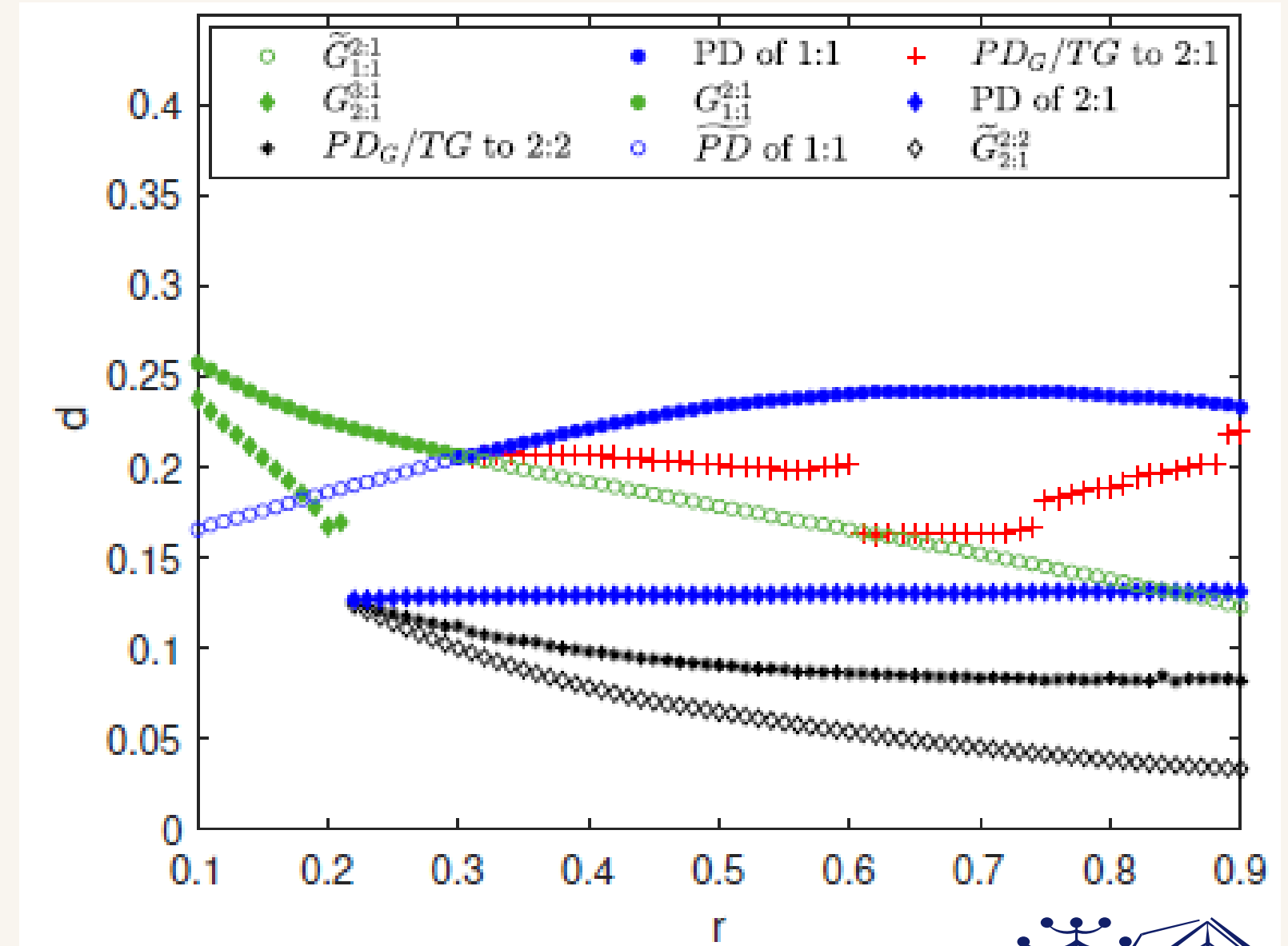
1:1/2T



$\beta=\pi/3$; reducing “s”

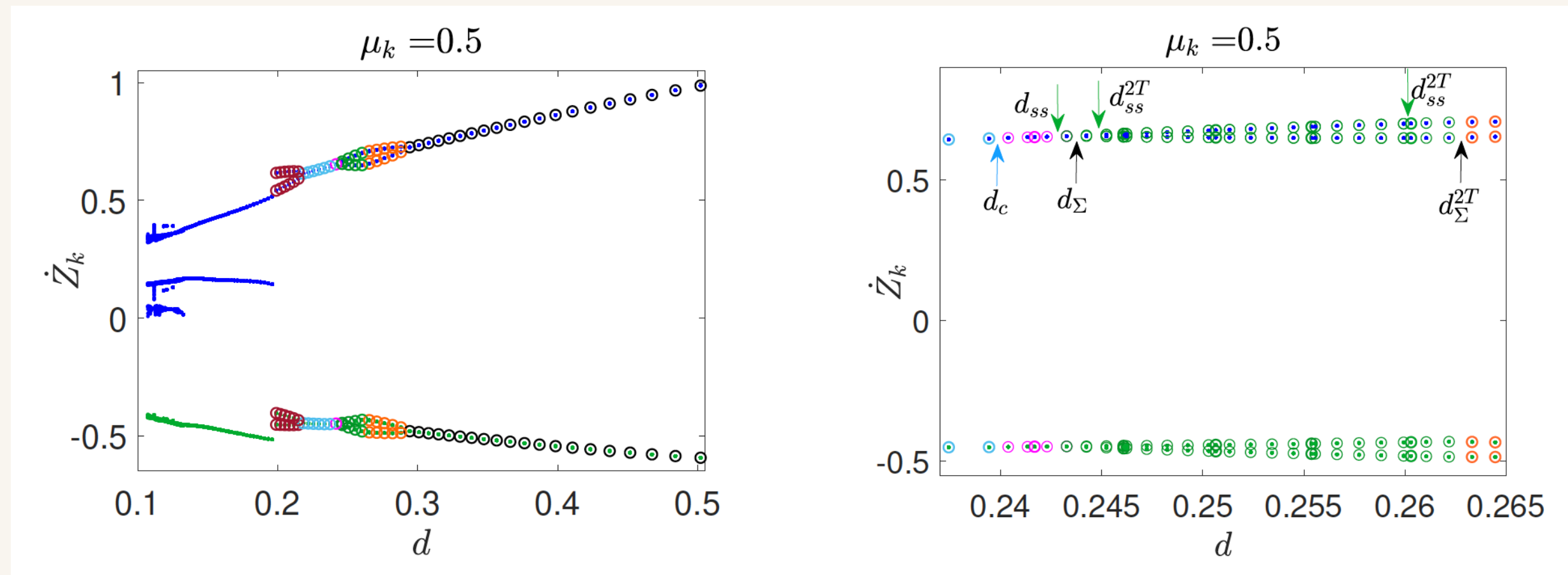
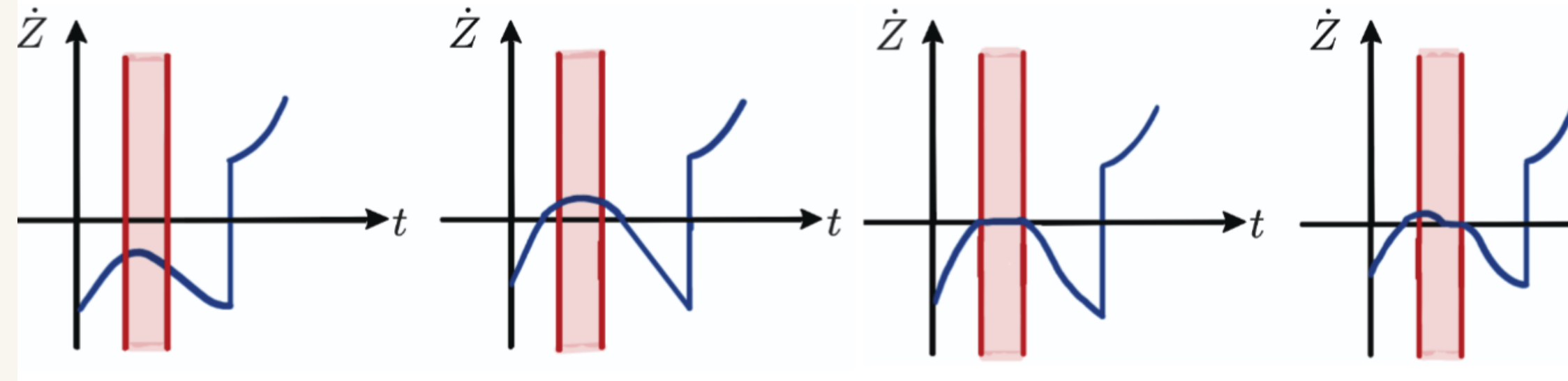
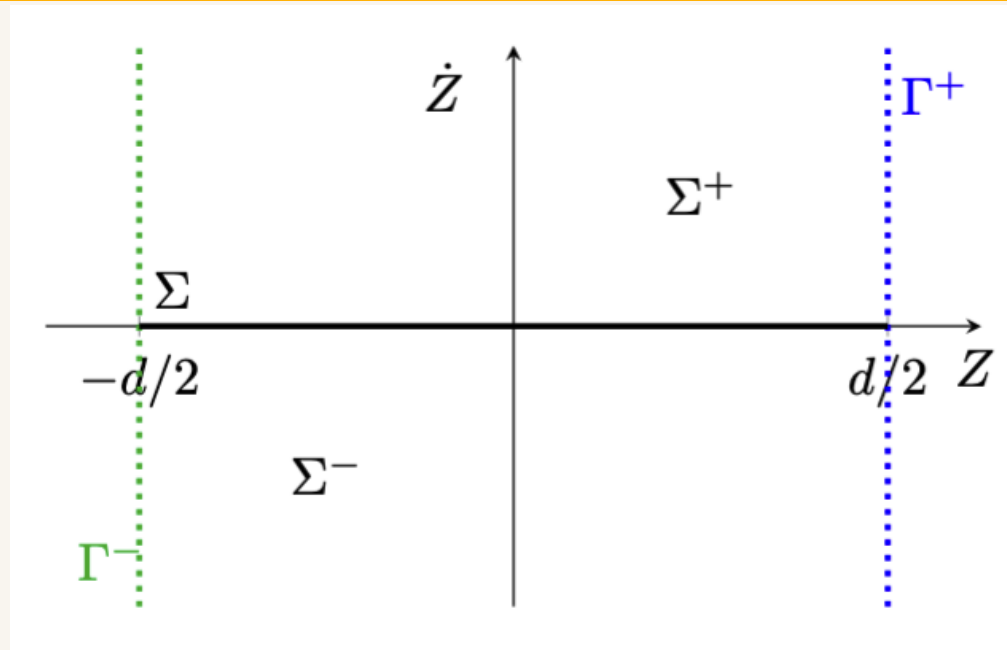


$\beta=\pi/6$; increasing “A”

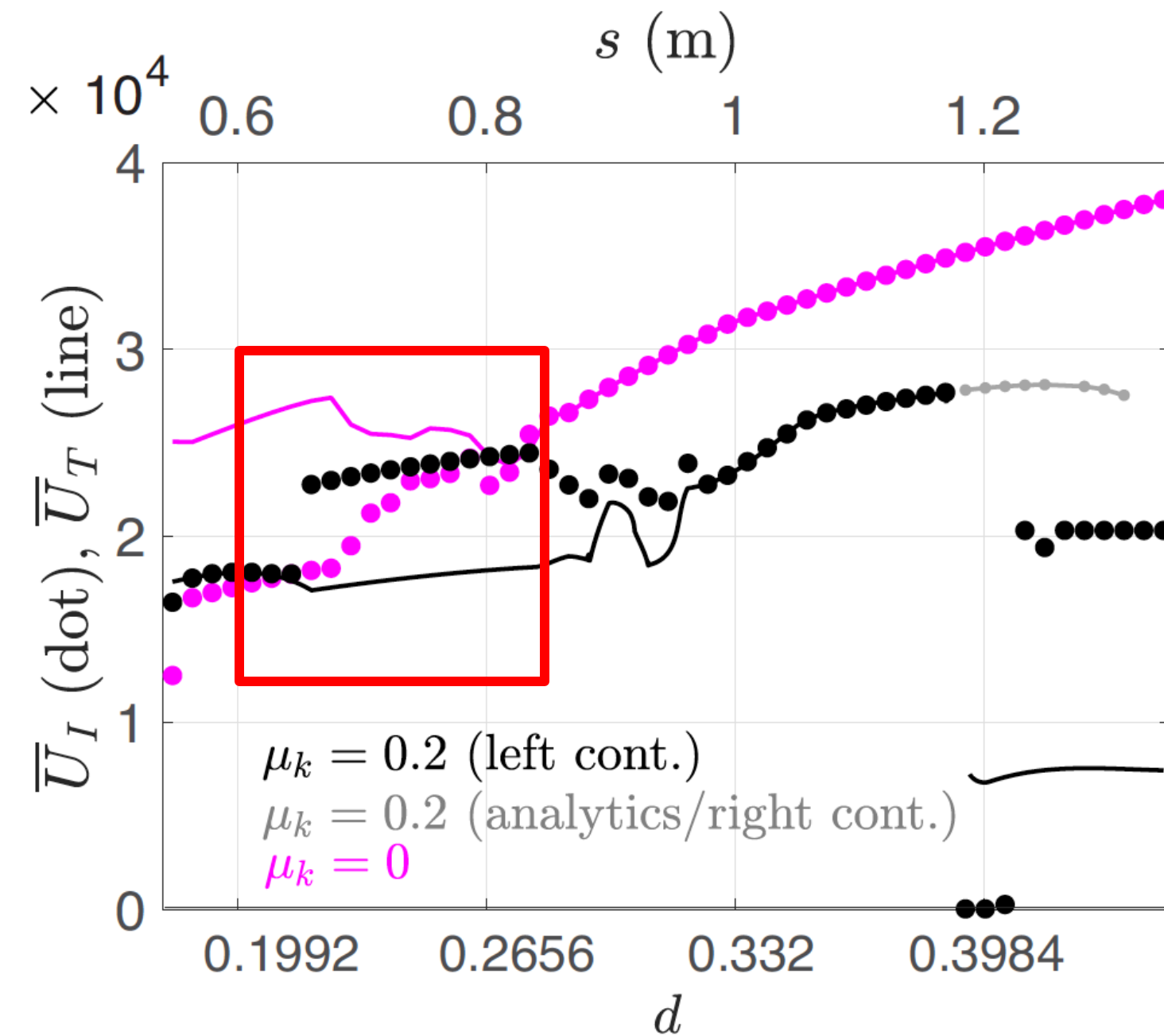
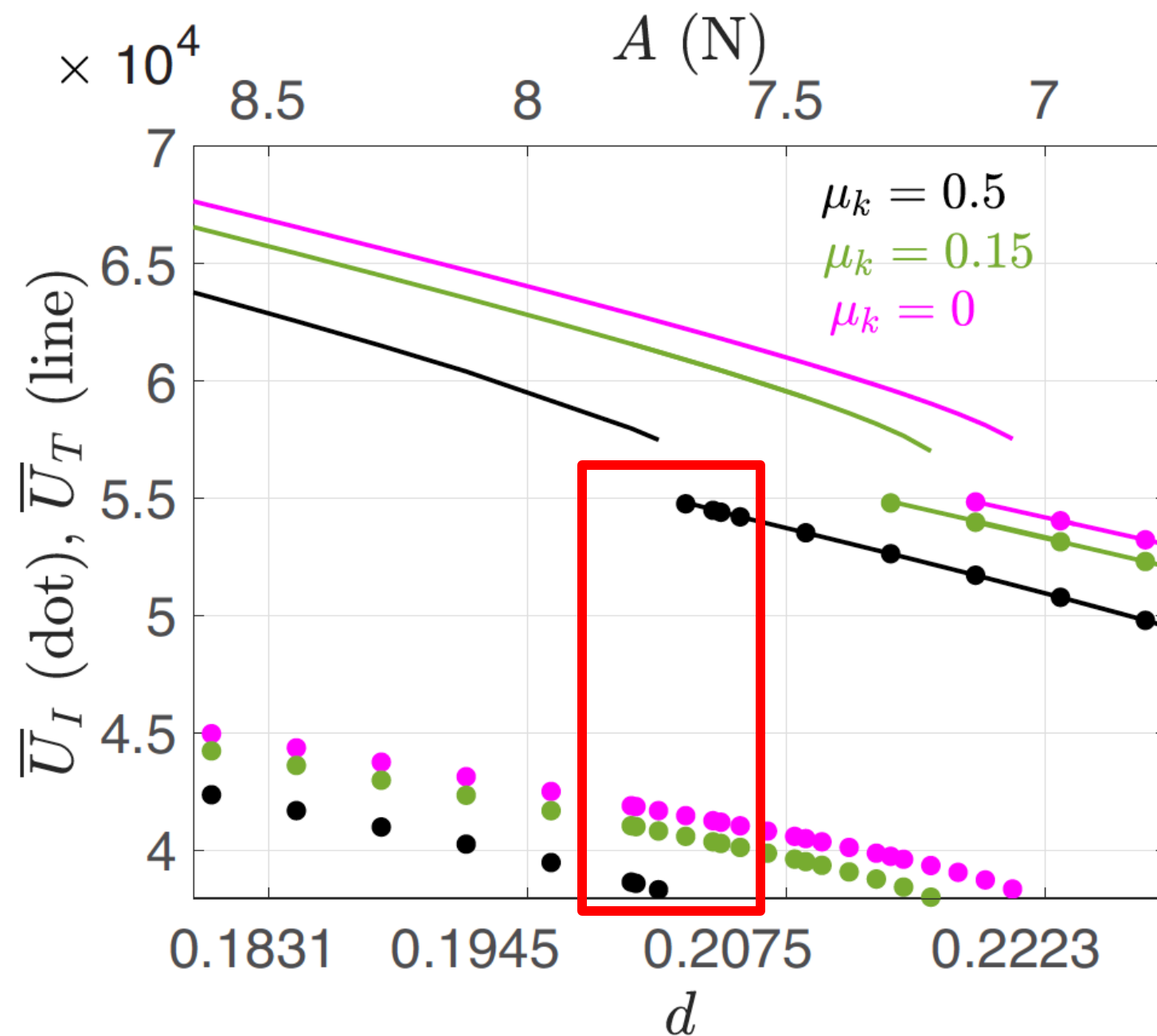


VI-CAPSULE with FRICTION

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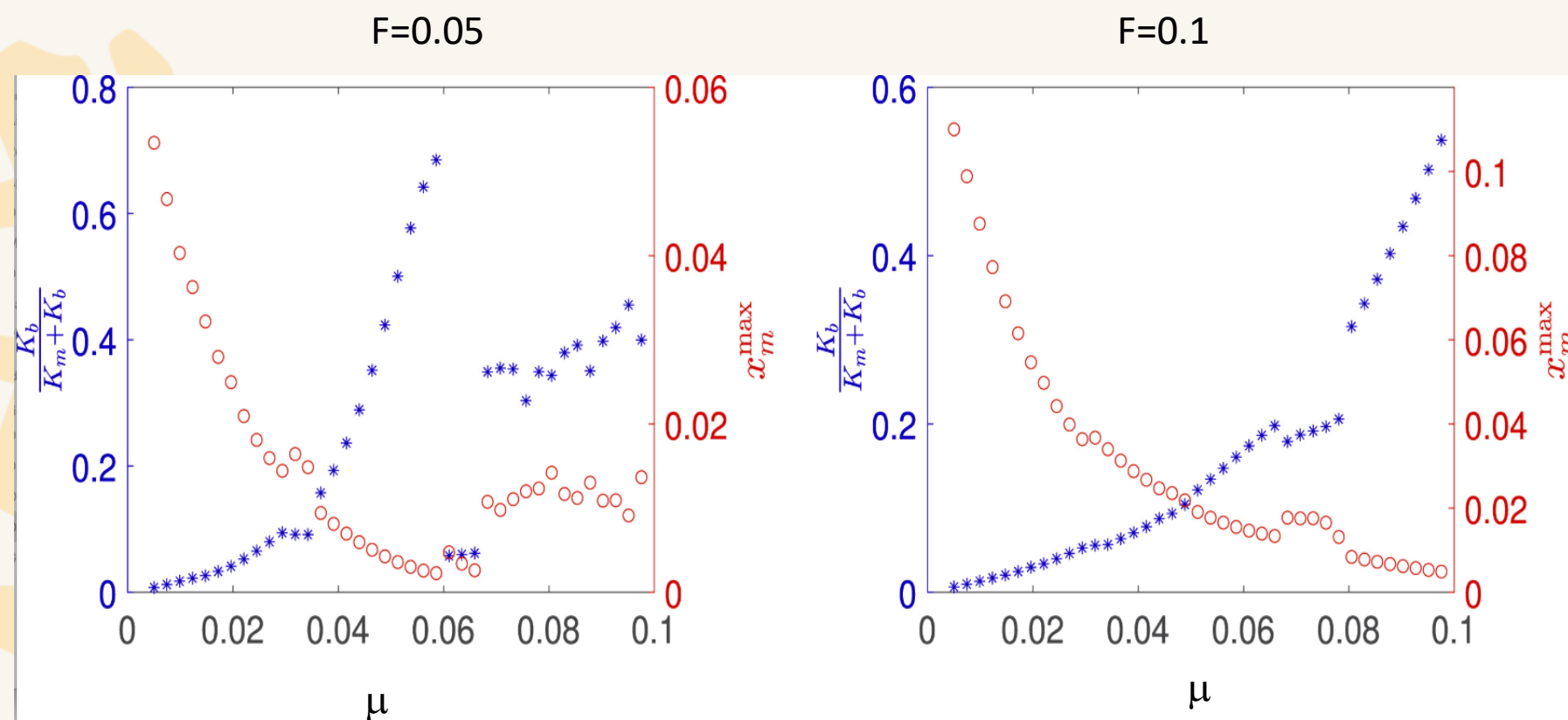
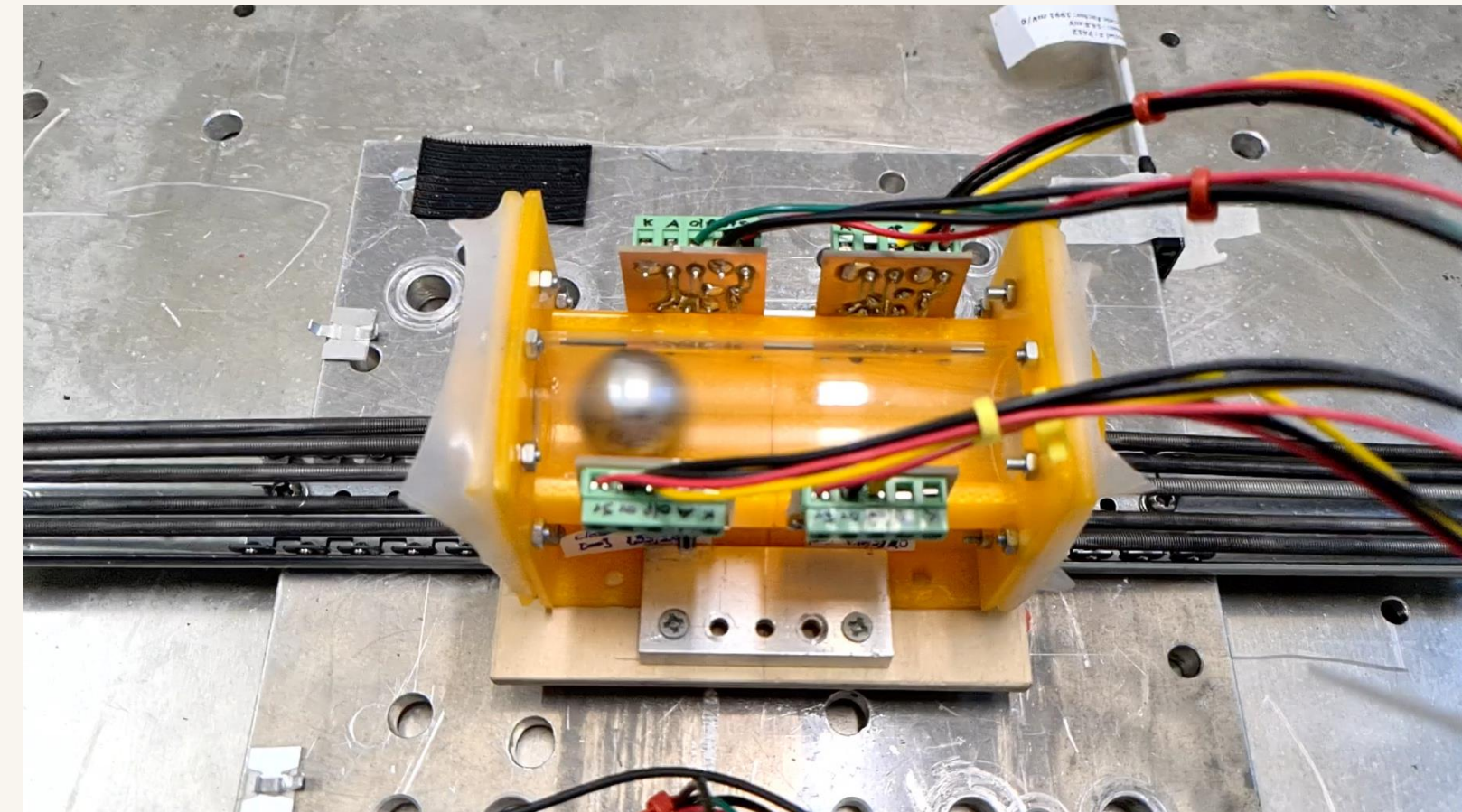
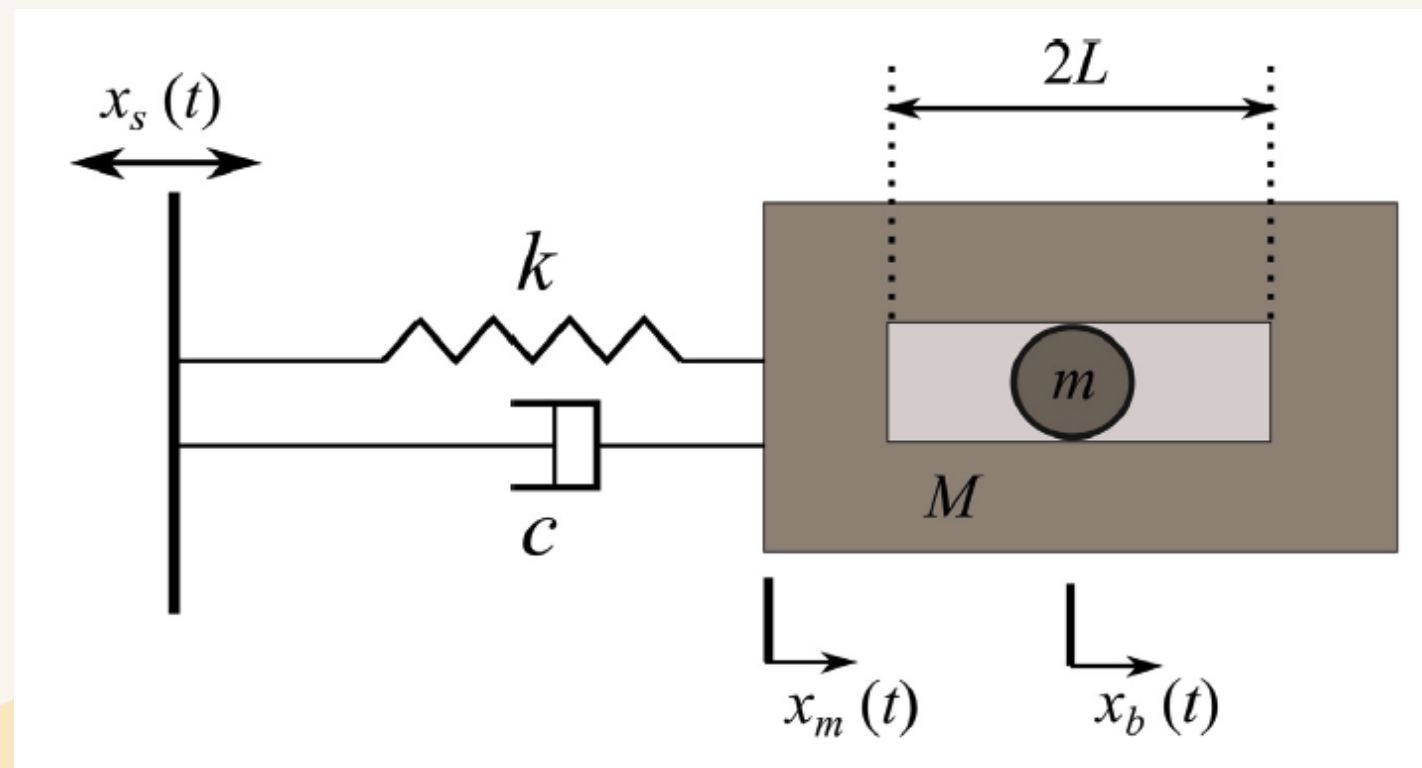
$$\begin{aligned}
 &1:1 \xrightarrow{PD} 1:1/2T \xrightarrow{d_{\Sigma}^{2T}} 1:1-1:1_s/2T \xrightarrow{d_{ss}^{2T}} 1:1-1:1_{cs}/2T \xrightarrow{d_{ss}^{2T}} 1:1-1:1_s/2T \xrightarrow{d_{\Sigma}} \\
 &1:1_s \xrightarrow{d_{ss}} 1:1_{cs} \xrightarrow{d_c} 1:1_c \xrightarrow{d_c^{2T}} 1:1_c/2T \xrightarrow{d_{\Gamma^+}} 2:1,
 \end{aligned}$$



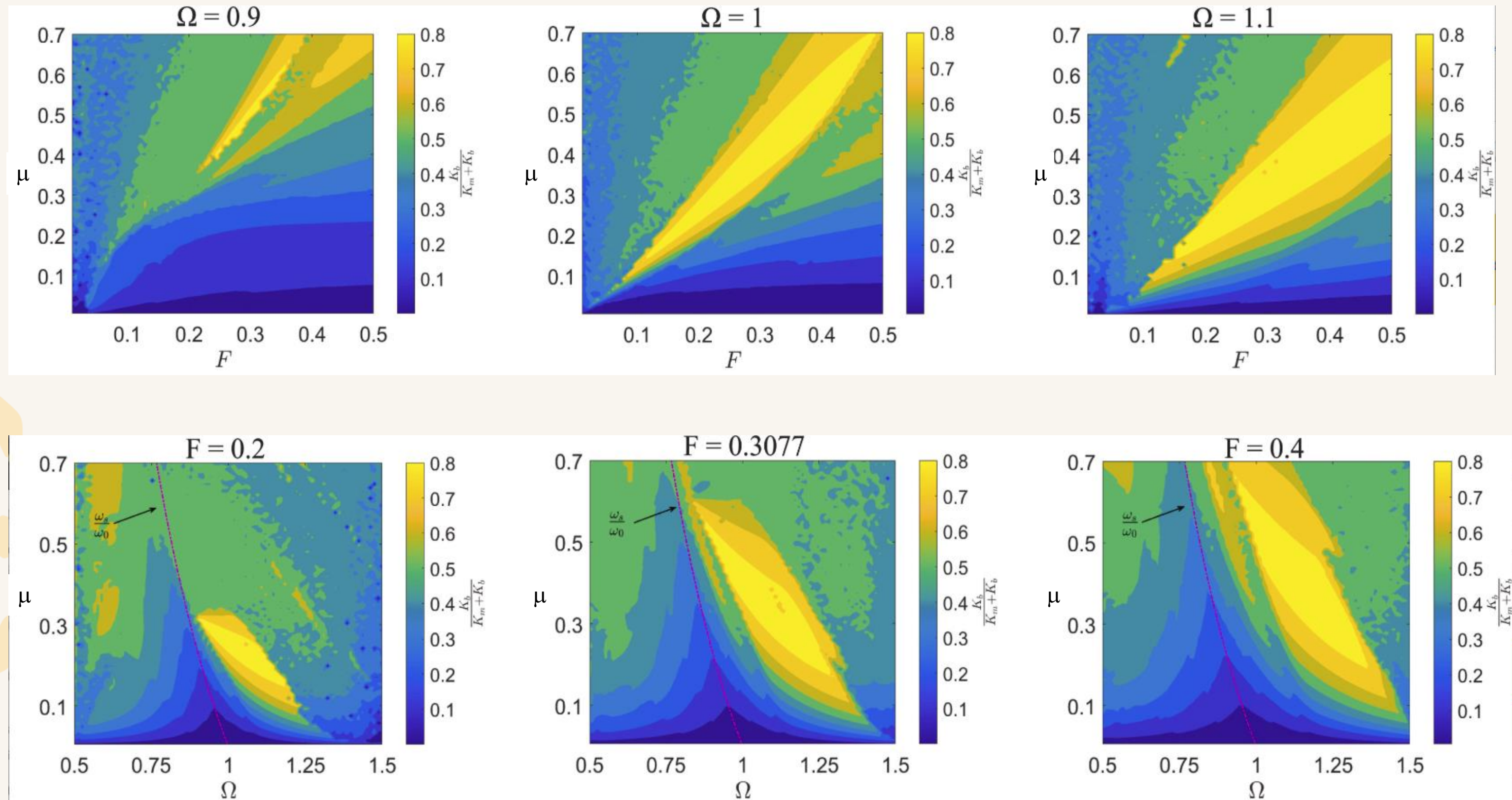
Targeted Energy Transfer

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Targeted Energy Transfer



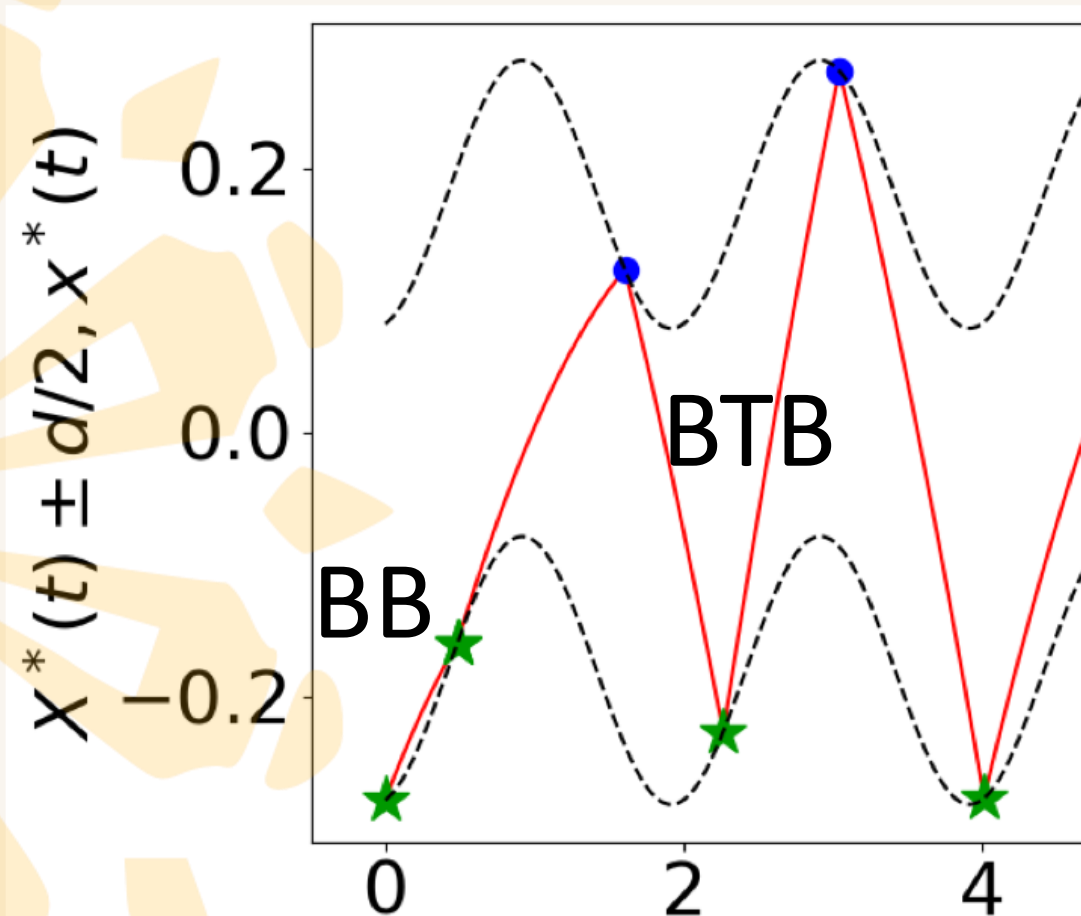
First return maps:

Use the short sequences of returns to ∂B return maps as the building blocks

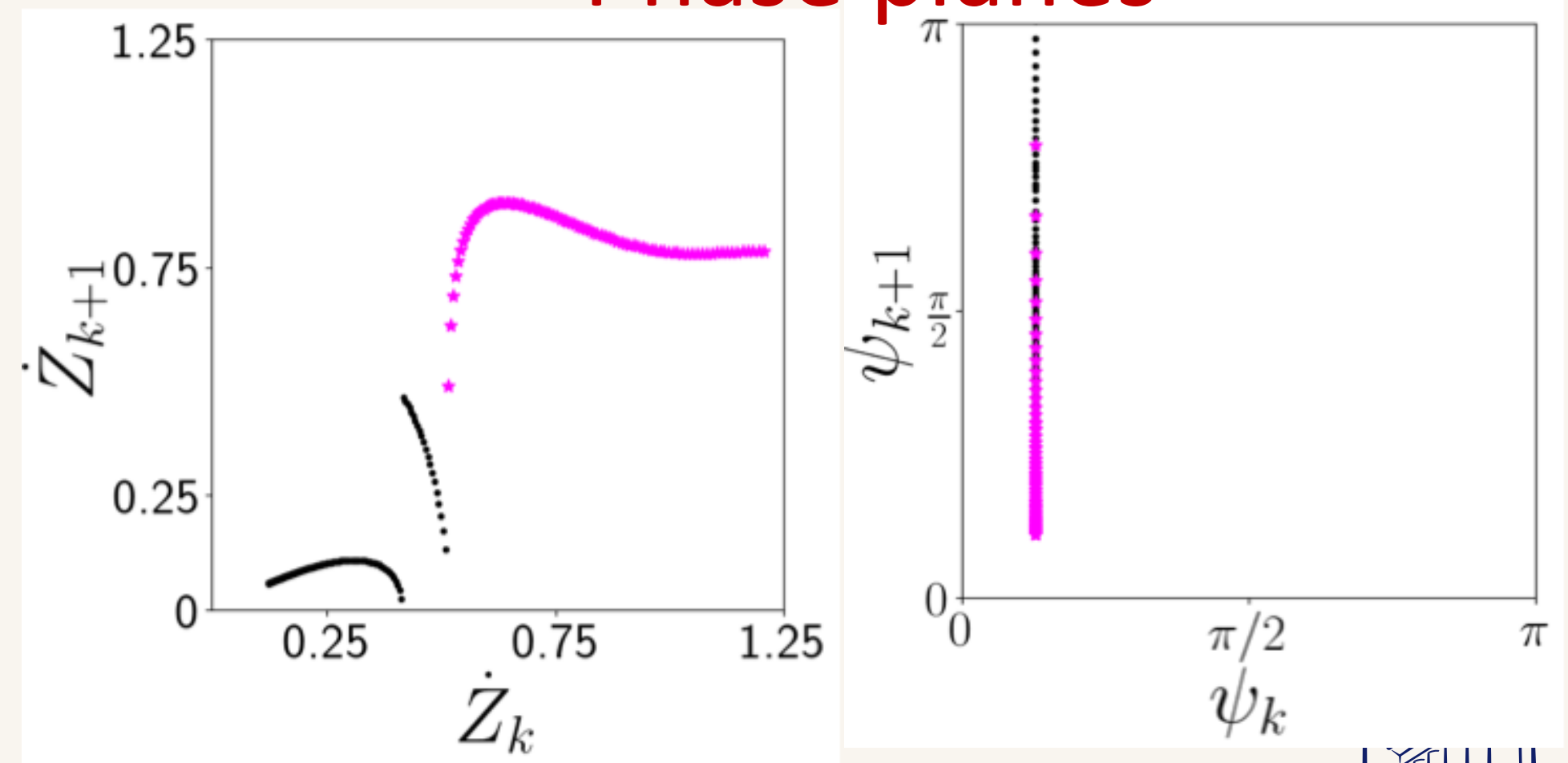
Over state space:

Sweep \dot{Z}_k fixed ψ_k

Time series

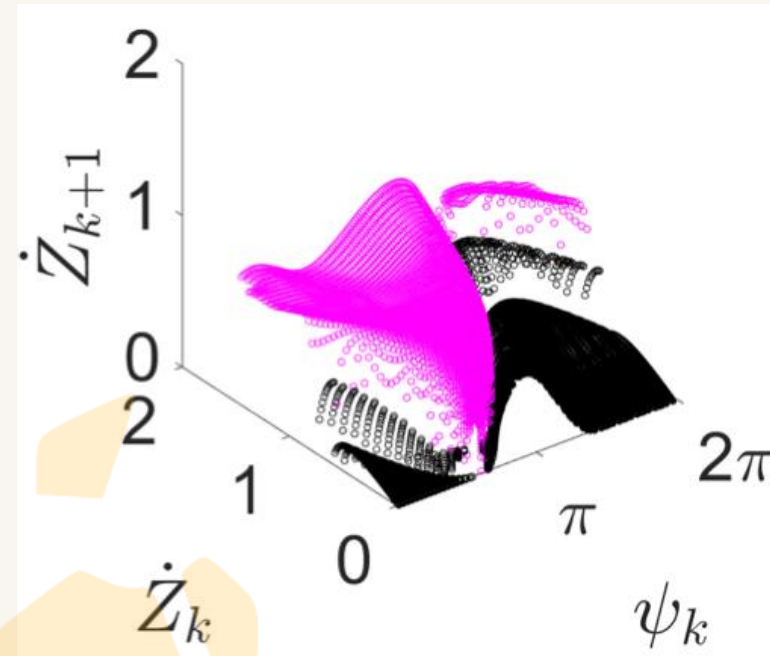


Phase planes

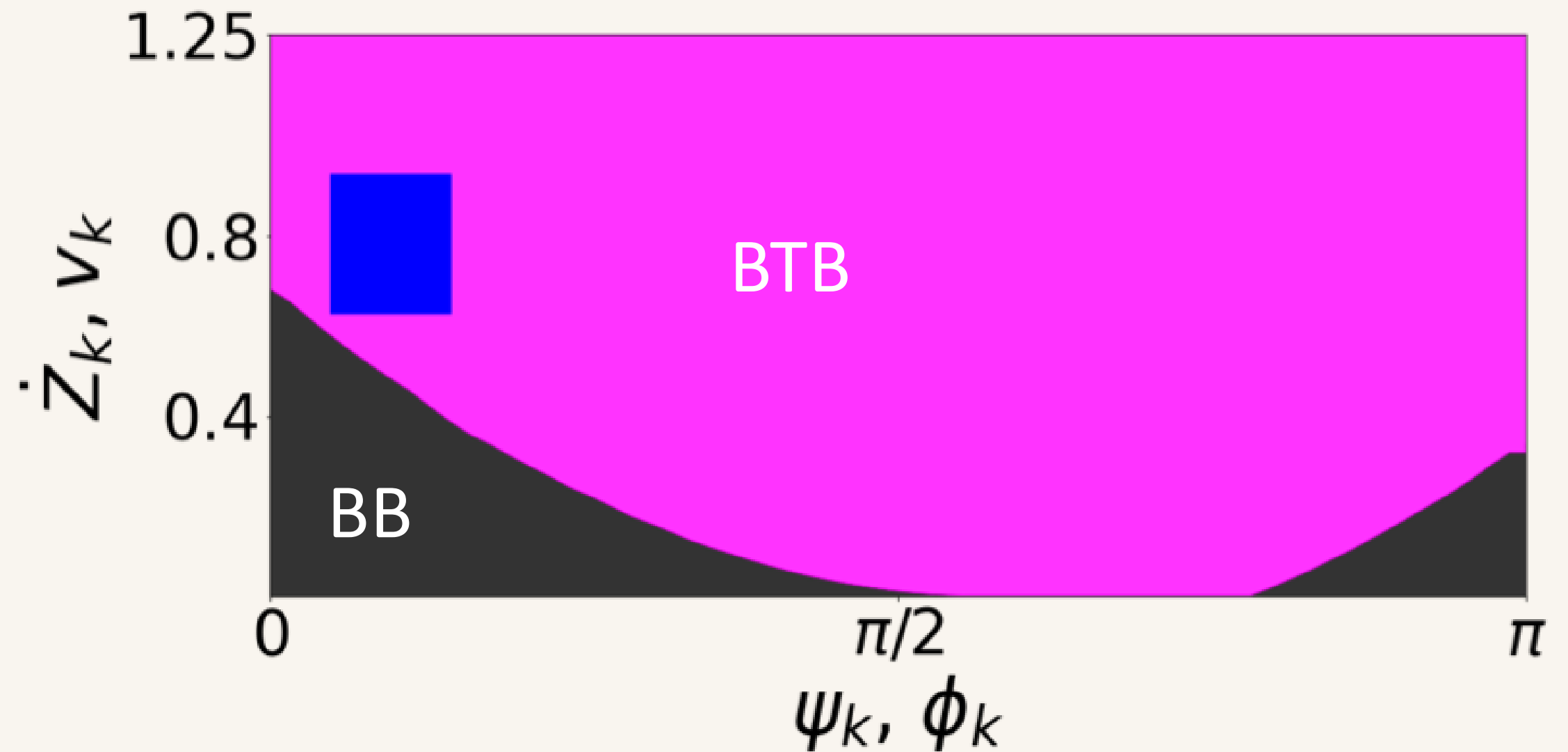
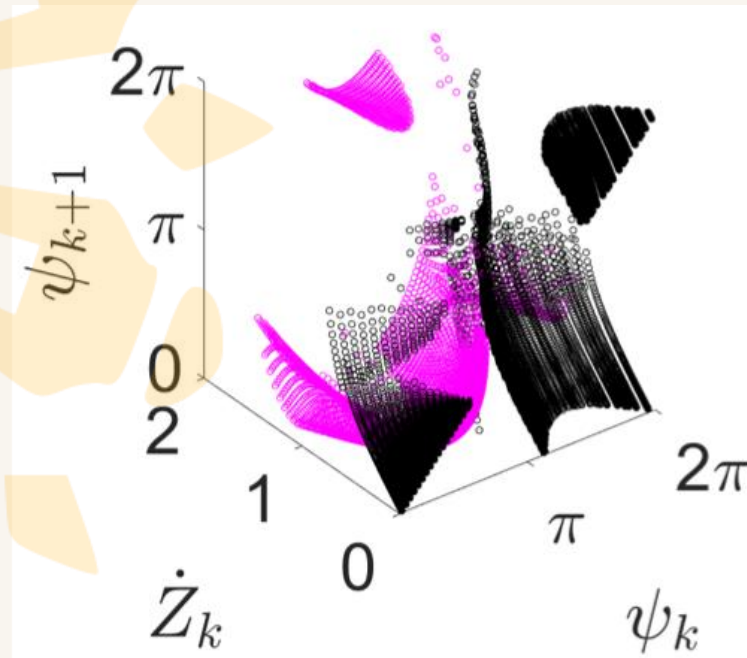


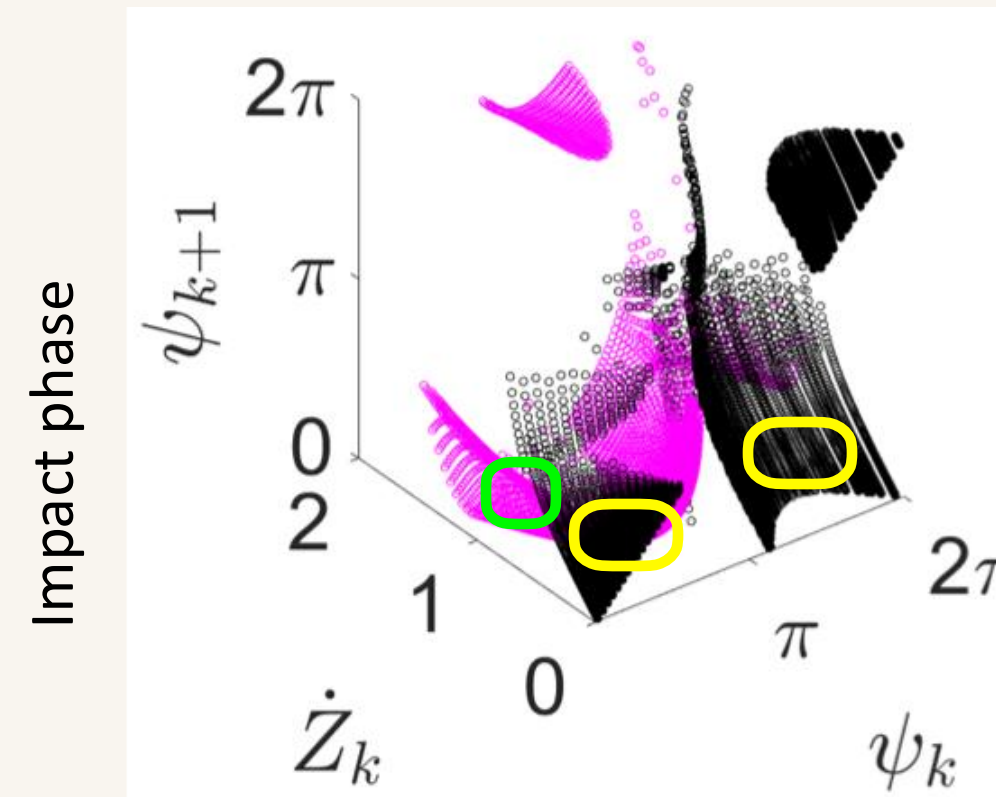
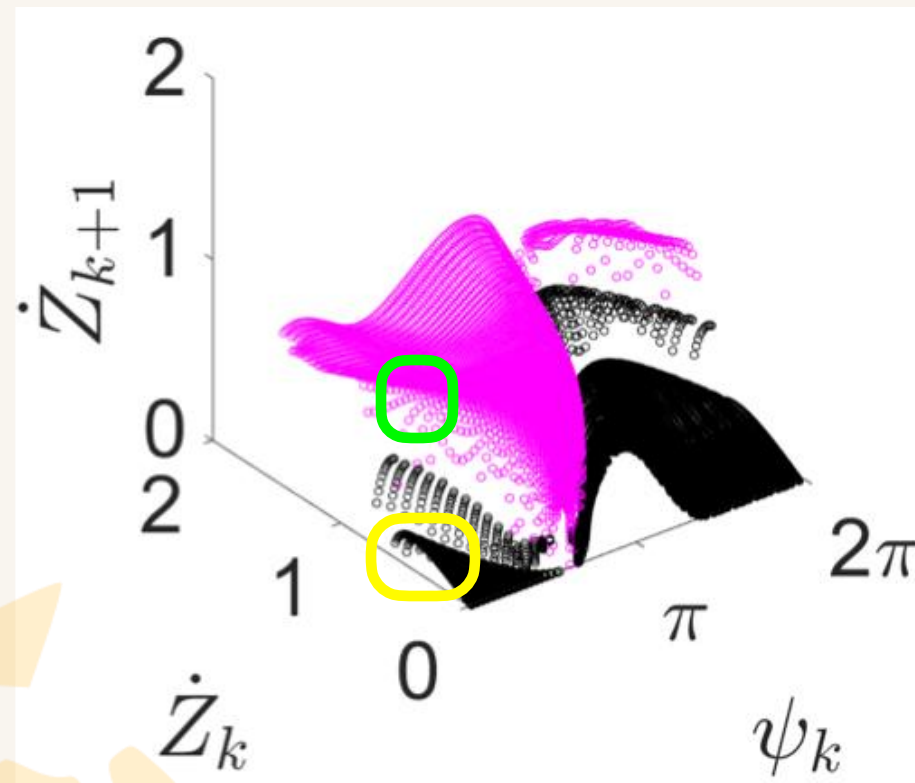
Return maps in Relative framework:

Impact velocity



Impact phase

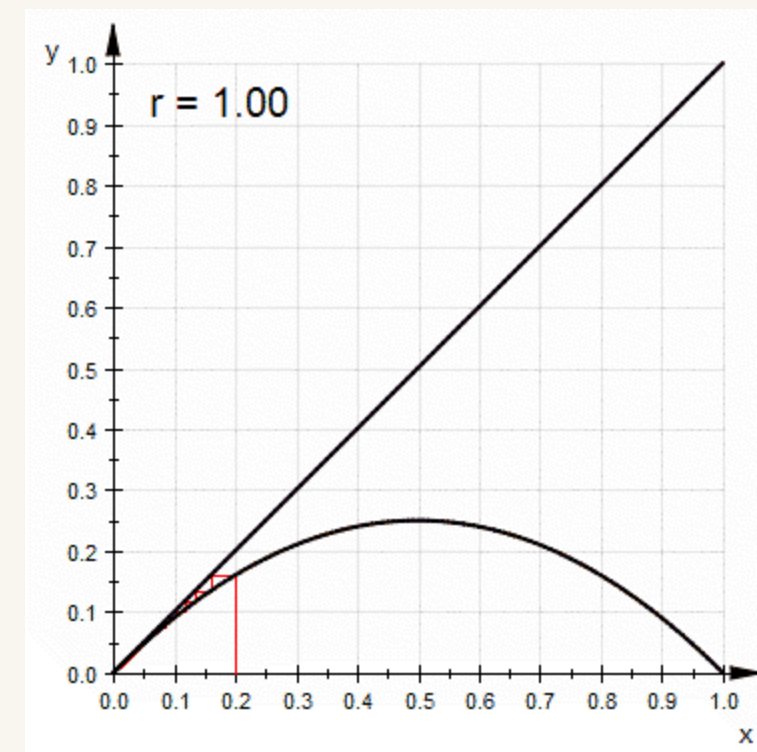
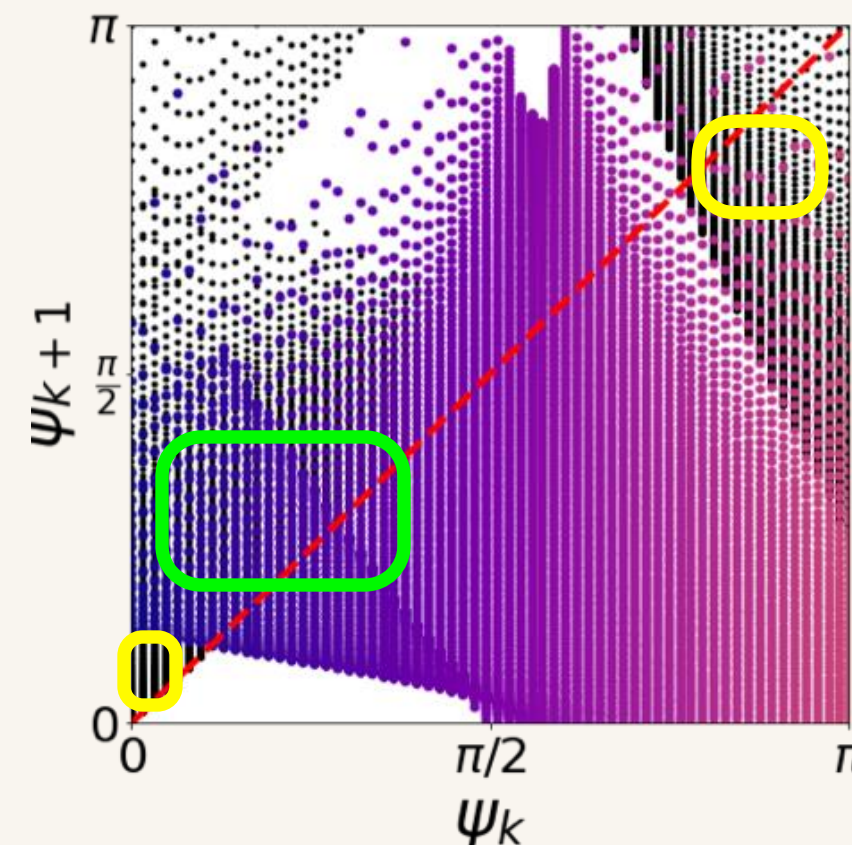
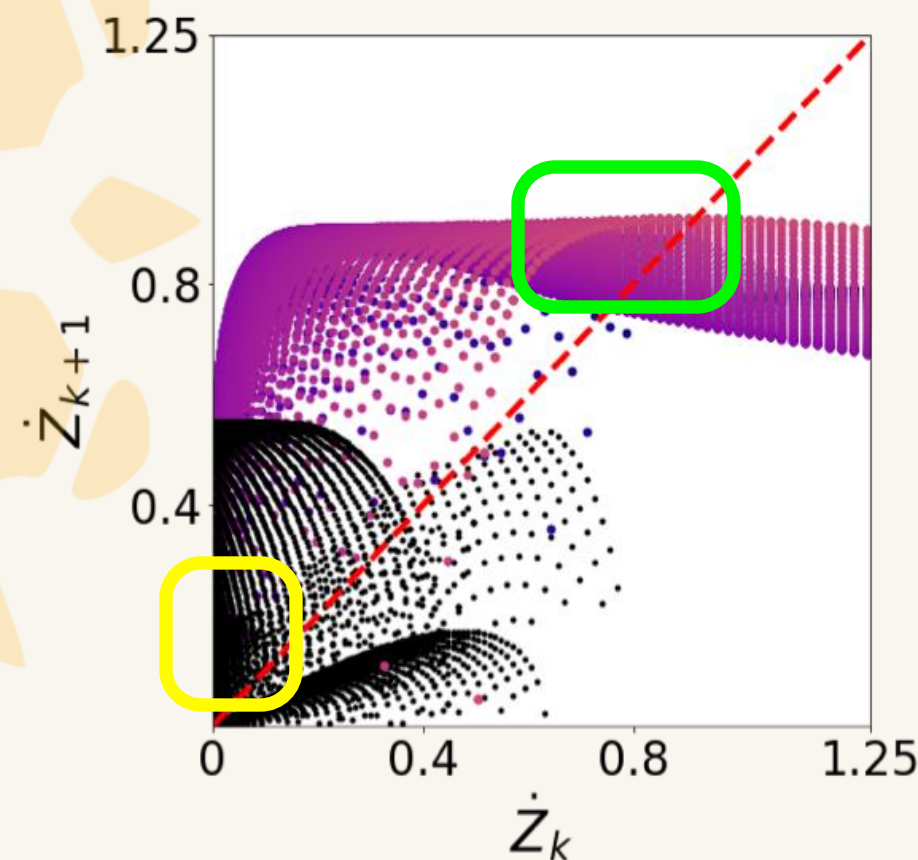




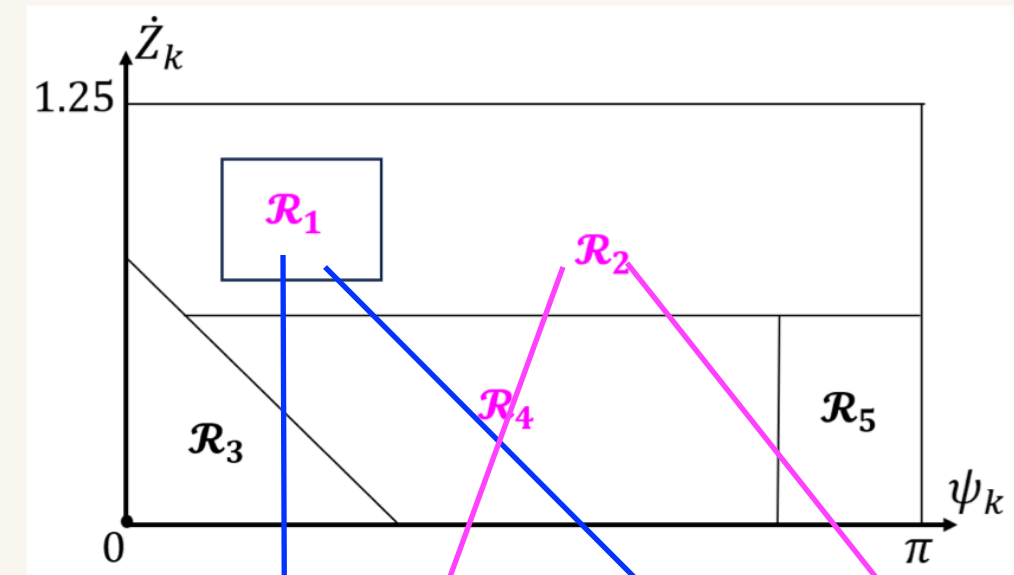
Exchange non-smooth impacts
for non-smooth surfaces

potential attracting
dynamics

potential transient
dynamics



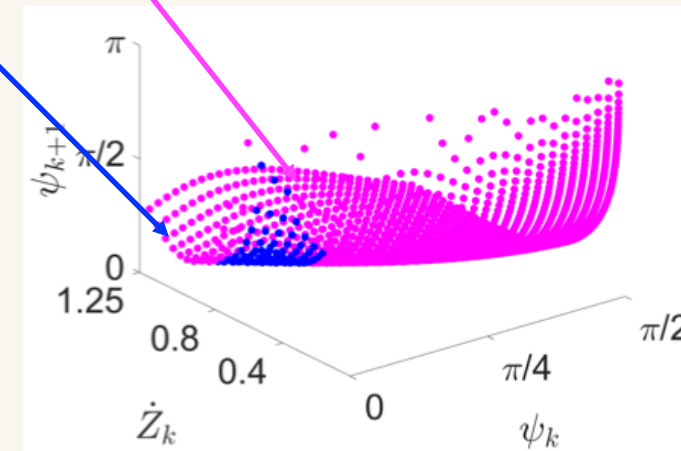
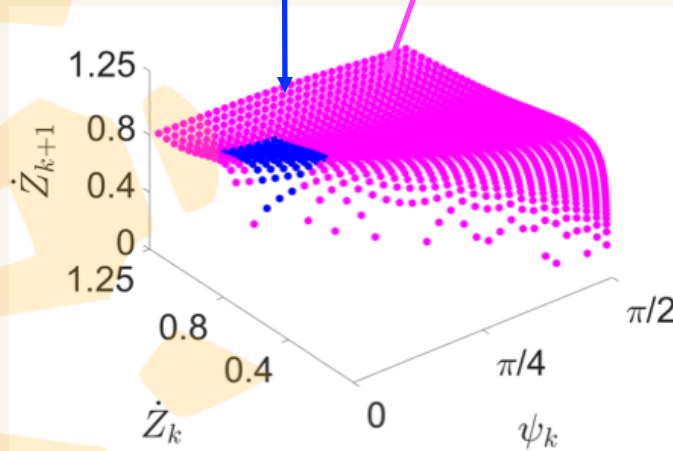
Cobweb
Logic map



\mathcal{R}_1

Region of likely return to same values of \dot{Z}_k, ψ_k in phase plane, return near the diagonal

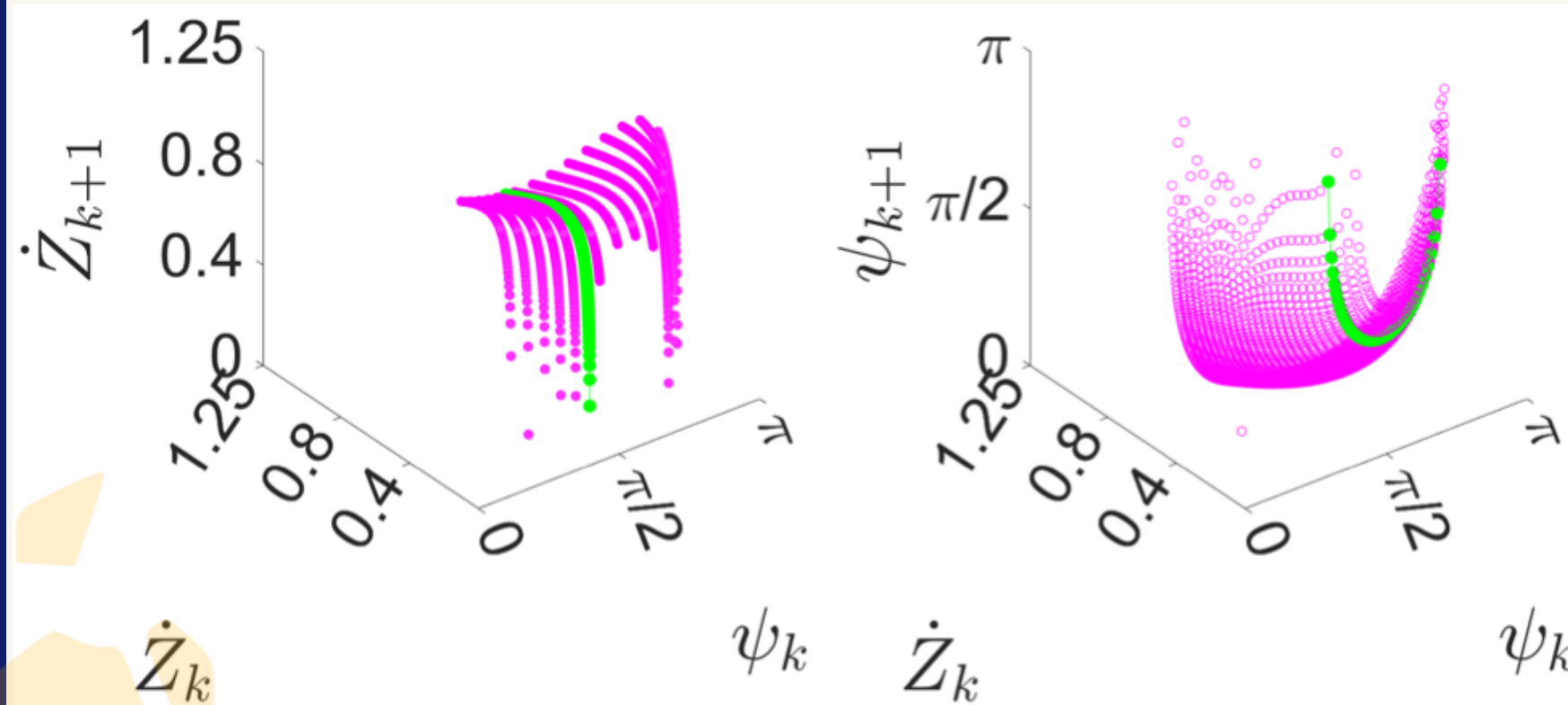
$$\begin{aligned} v_{k+1}(v_k, \phi_k) &= f_1(v_k, \phi_k) = \\ & b_0 + b_1\phi_k + b_2v_k + b_3\phi_k^2 + b_4\phi_kv_k + b_5v_k^2 + b_6\phi_k^2v_k + b_7\phi_kv_k^2 + b_8v_k^3, \\ \phi_{k+1}(v_k, \phi_k) &= g_1(v_k, \phi_k) = \\ & a_0 + a_1\phi_k + a_2v_k + a_3\phi_k^2 + a_4\phi_kv_k + a_5v_k^2 + a_6\phi_k^2v_k + a_7\phi_kv_k^2 + a_8v_k^3. \end{aligned}$$



$$\mathcal{R}_{1.2}(d) = \left\{ (\dot{Z}_k, \psi_k) : \frac{1}{\delta} < \left| \frac{\psi_{k+1}}{\psi_k} \right| < \delta \text{ and } \frac{1}{\delta} < \left| \frac{\dot{Z}_{k+1}}{\dot{Z}_k} \right| < \delta \right\}$$

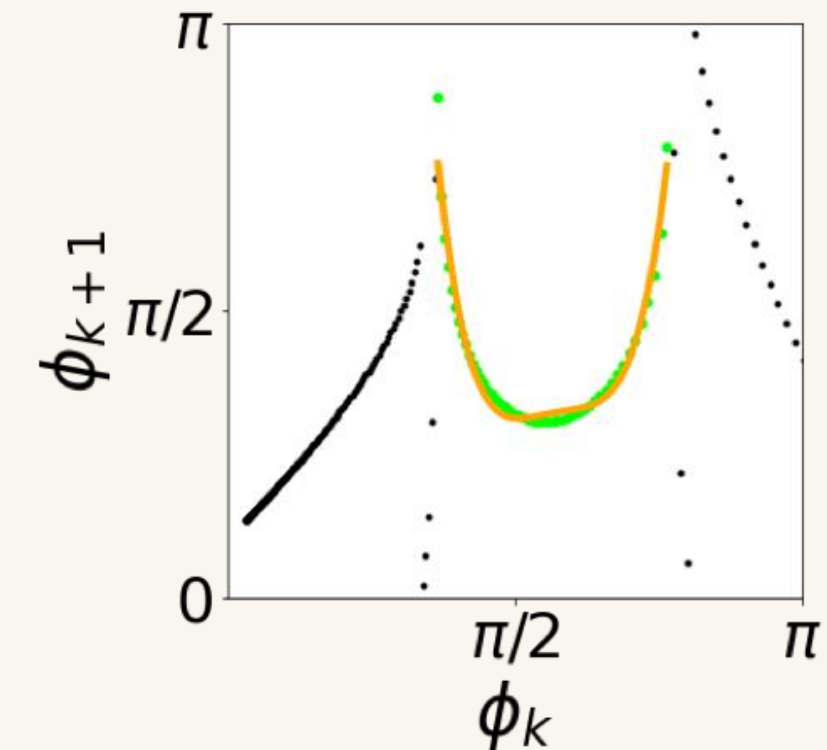
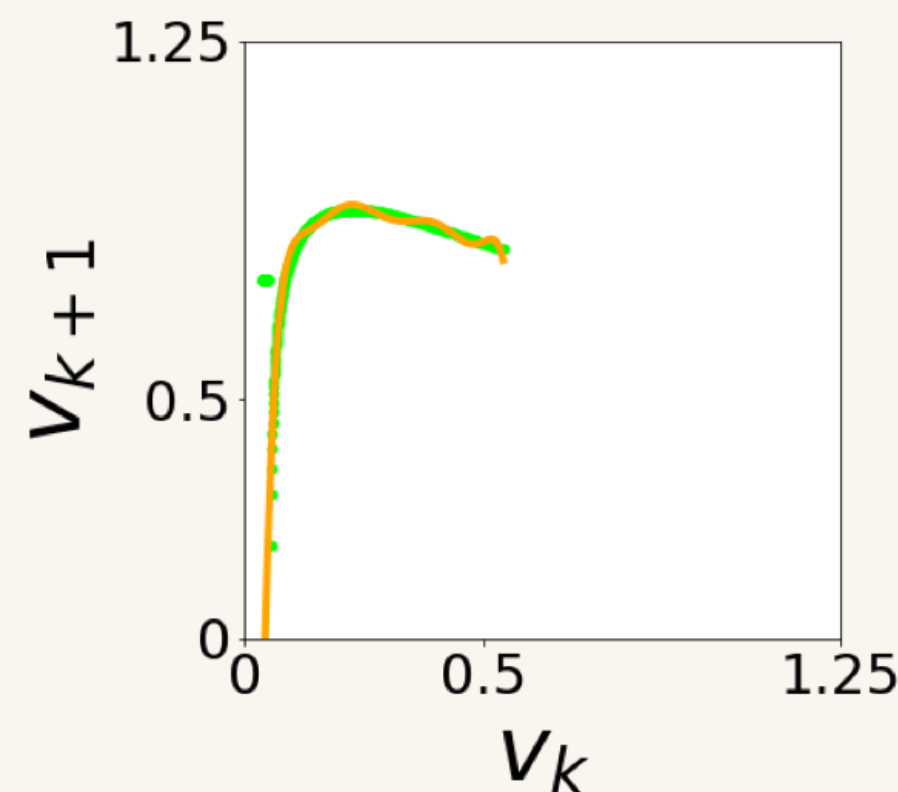
Remainder of the region is transient BTB behavior: shape of surface and/or not near diagonal in phase space

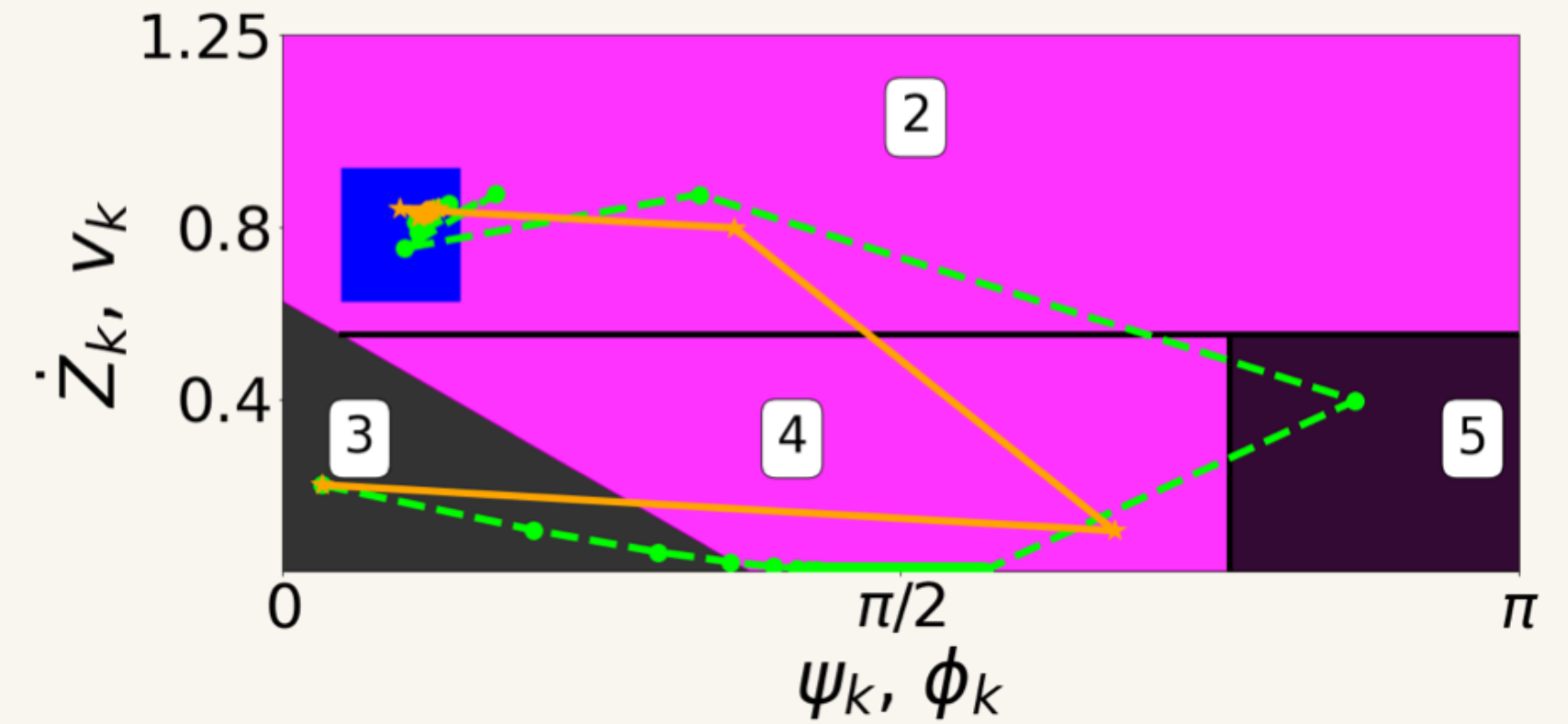
Approximate separable 1D maps in transient regions (R2, R4, R5):



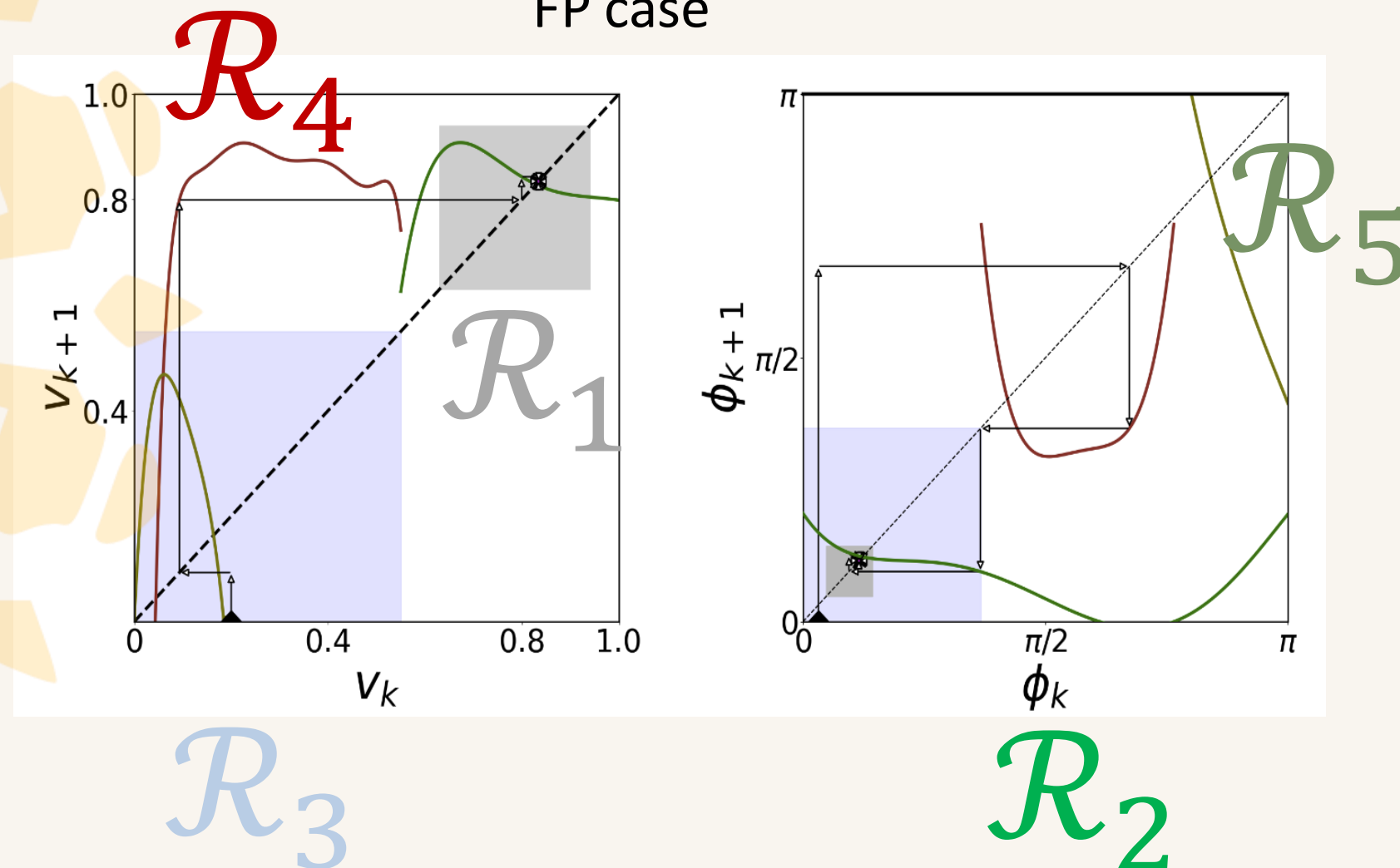
$$v_{k+1}(v_k) = f_2(v_k) = b_{20}v_k^5 + b_{21}v_k^4 + b_{22}v_k^3 + b_{23}v_k^2 + b_{24}v_k + b_{25},$$

$$\phi_{k+1}(\phi_k) = g_2(\phi_k) = a_{20}\phi_k^5 + a_{21}\phi_k^4 + a_{22}\phi_k^3 + a_{23}\phi_k^2 + a_{24}\phi_k + a_{25}.$$





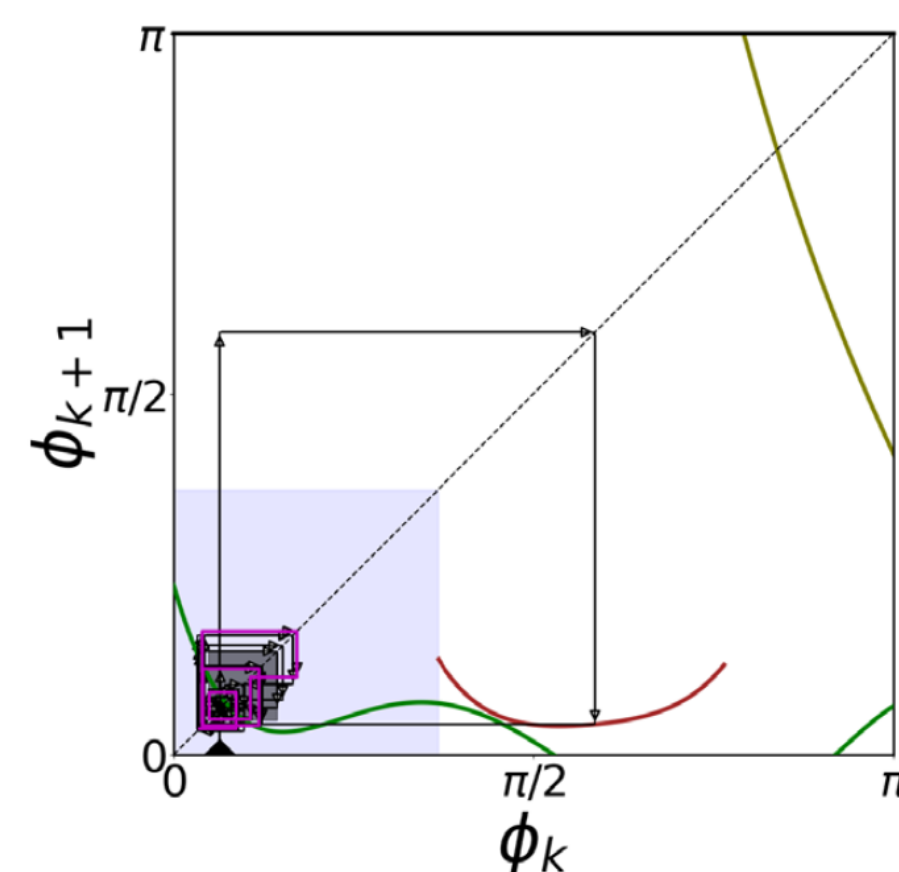
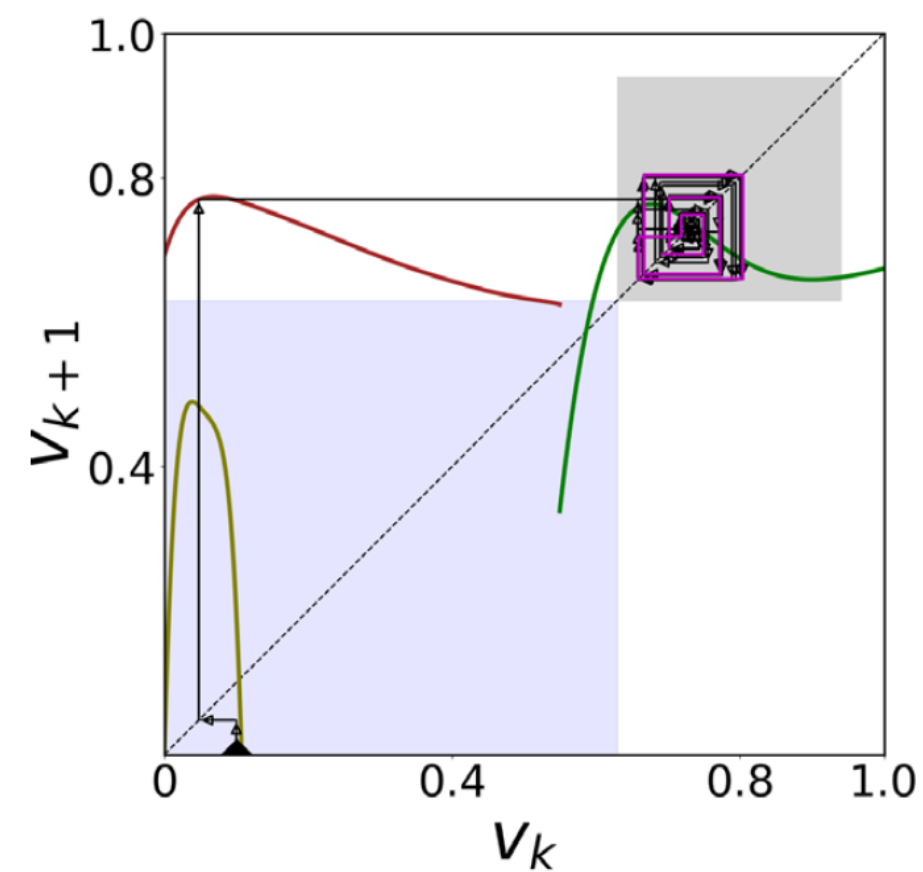
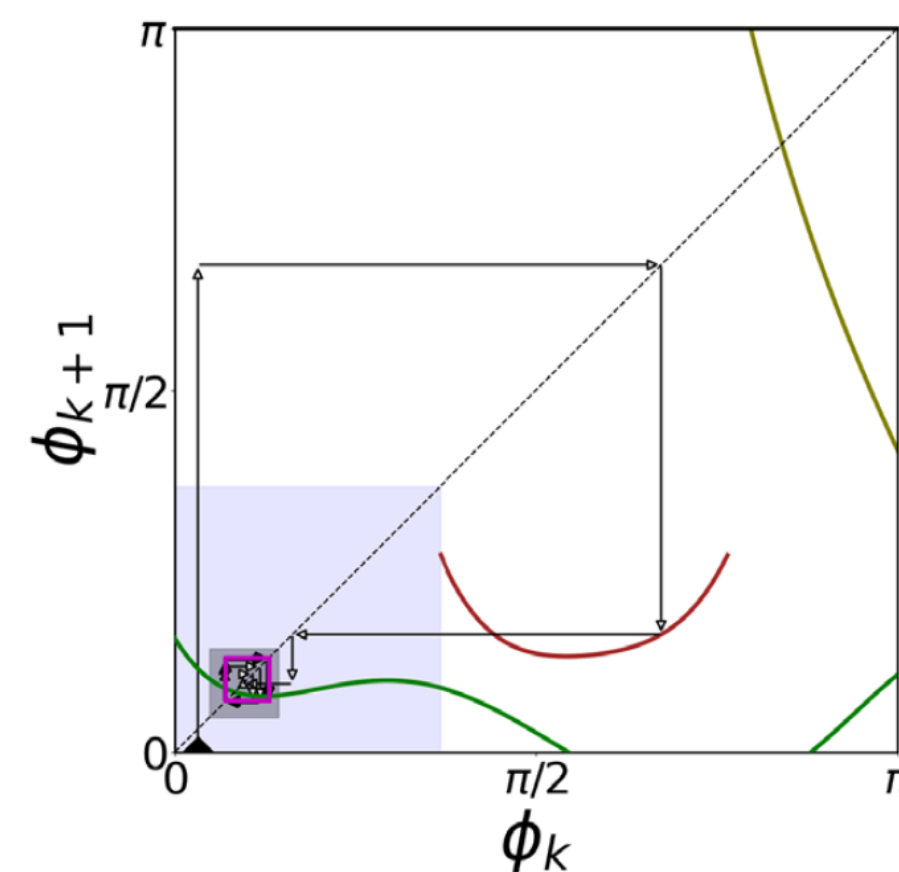
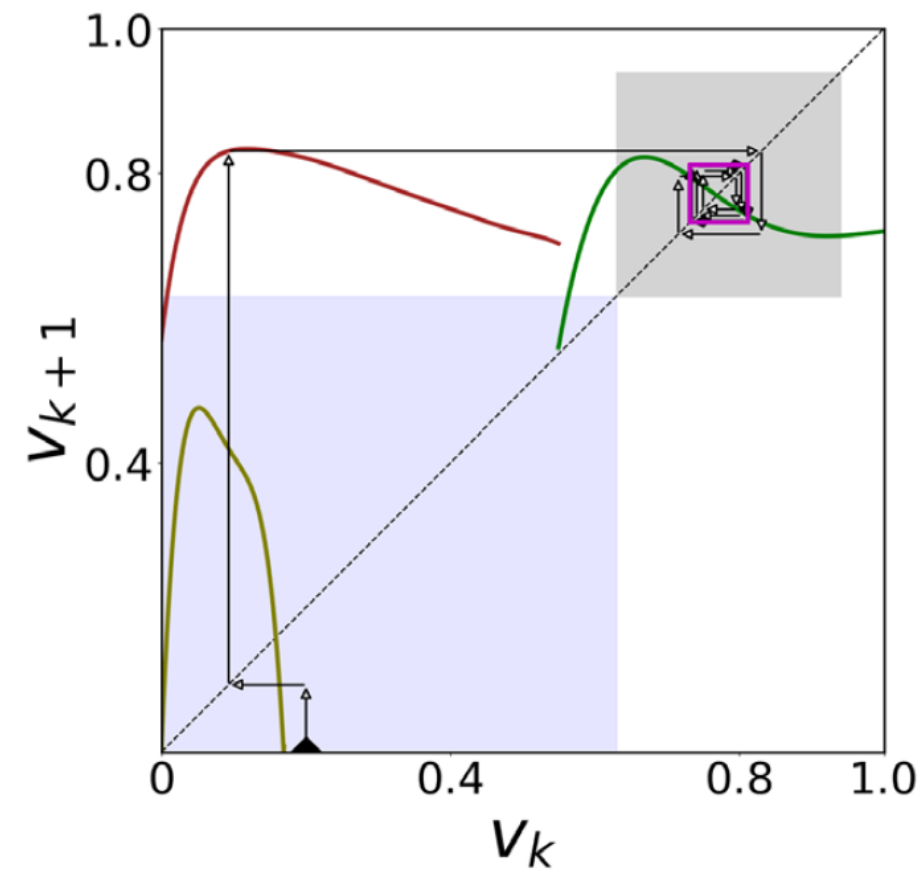
FP case

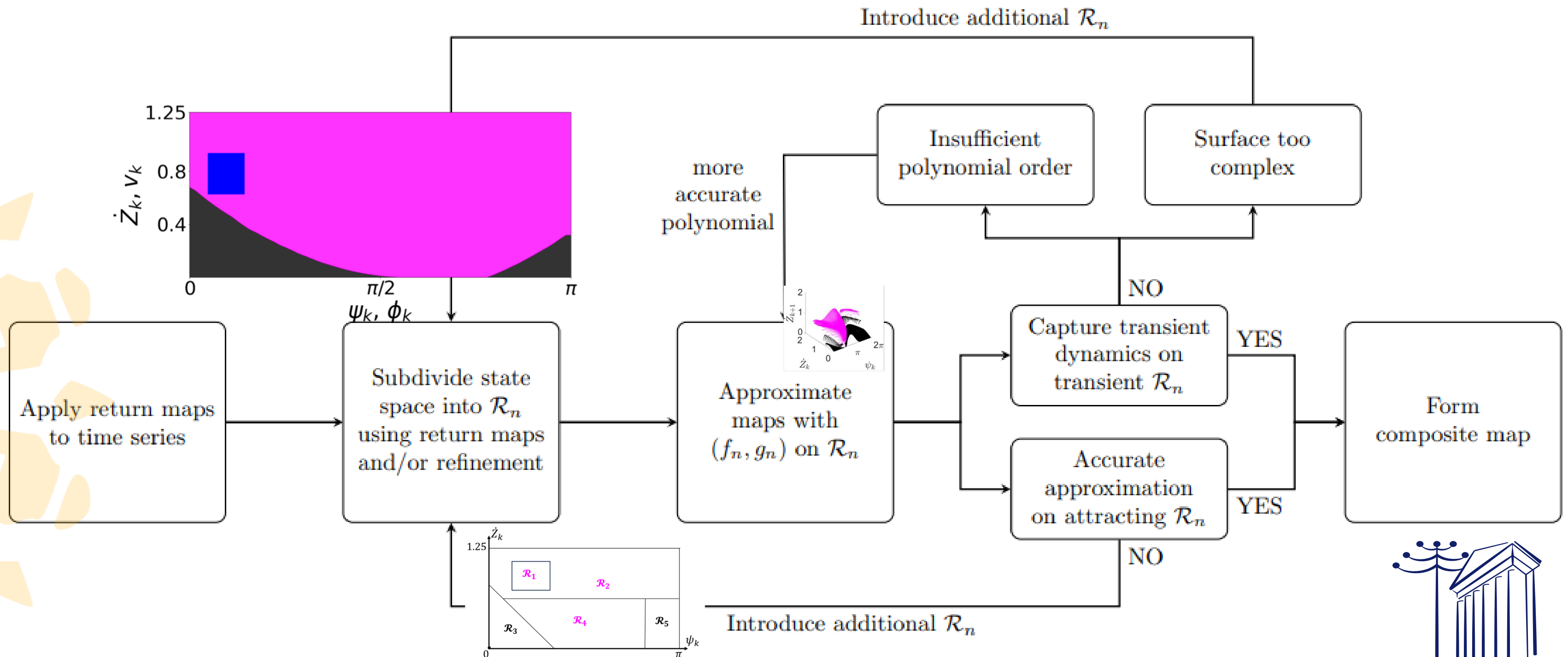


Computer-assisted method

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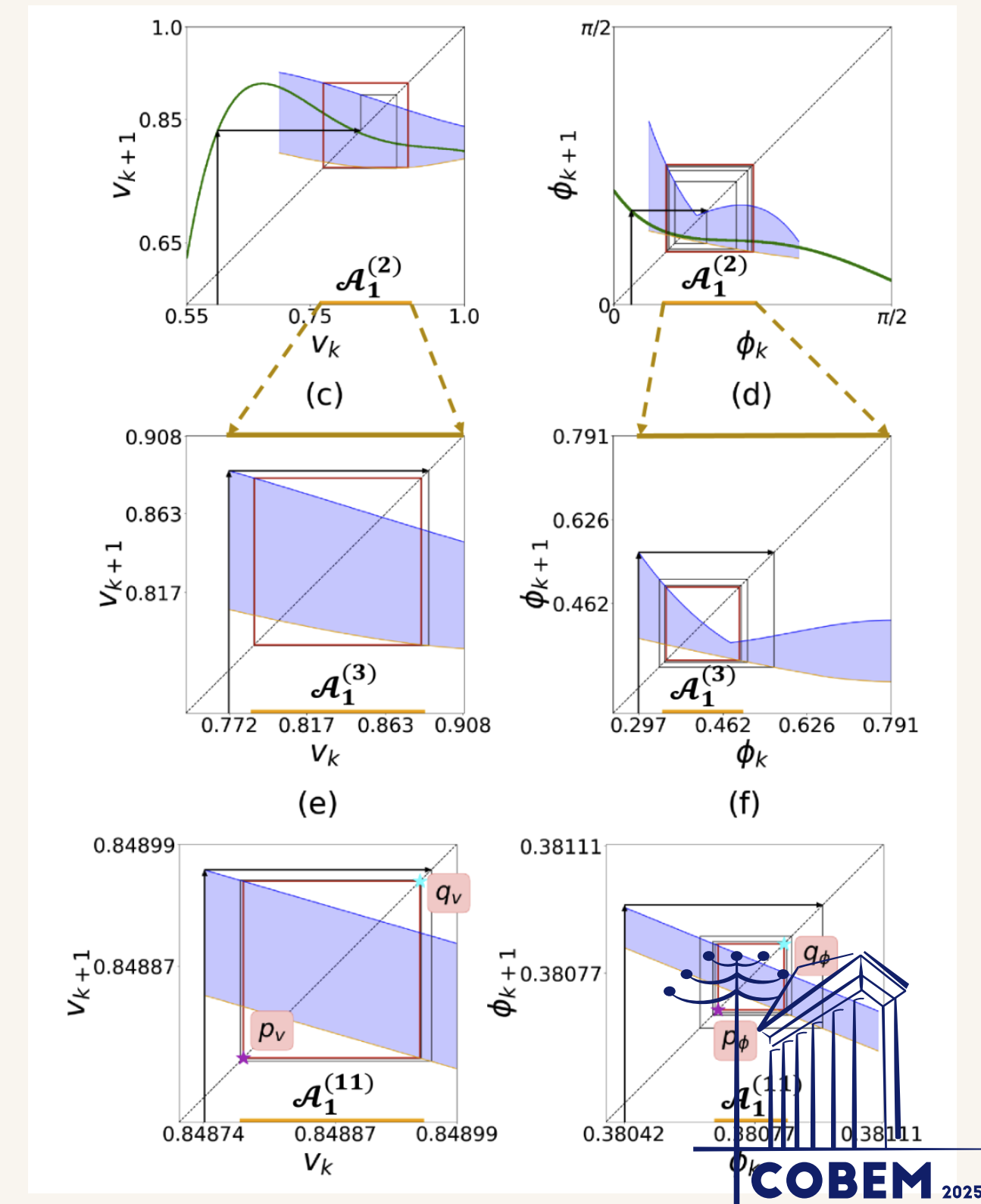
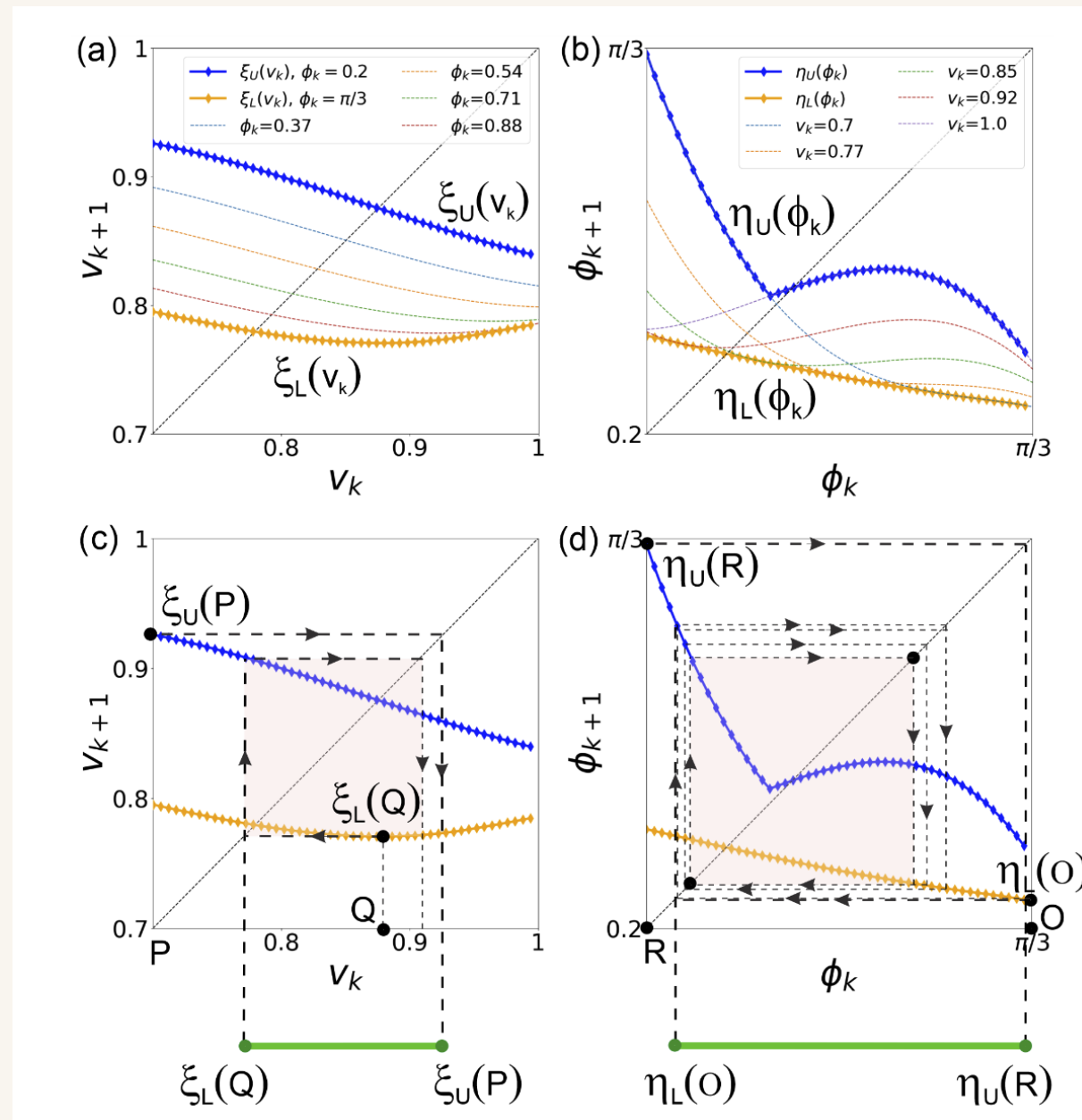
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Worst Case Scenario:

$$\eta_n^{(N)} = \begin{cases} \phi_{k+1} = \eta_{\max}^{(N)}(\phi_k), & \text{where } \eta_{\max}^{(N)} := \max_{v_k \in \mathcal{A}_n^{(N)}} \{g_n(v_k, \phi_k)\}, \\ \phi_{k+1} = \eta_{\min}^{(N)}(\phi_k), & \text{where } \eta_{\min}^{(N)} := \min_{v_k \in \mathcal{A}_n^{(N)}} \{g_n(v_k, \phi_k)\}. \end{cases}$$



Current State of the art

- Some substantial progress has been made in understanding of non-smooth system, including VI systems
- Almost anything is possible in such systems in terms of nonlinear behavior and bifurcations, including ghost bifurcations
- Global stability analysis can deal with low dimensional systems
- TET can be effective not only near the resonance, but also force should match the mass ratio for optimal performance
- Energy harvesting and Vibration Mitigation are not exactly two sides of one coin

Challenges

- Large systems require high computational power tracking multiple impacts
- Automate the process of deriving maps
- Global stability analysis for MDOF systems
- Design systems with prescribed behavior (energy localization, energy transfer, etc)

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<https://sites.google.com/view/vienergyharvest>

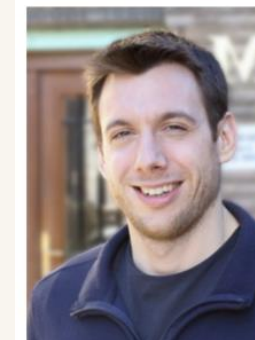
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QUESTIONS

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